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Enhancement of quasi-static compression strength for aluminum closed cell foam blocks shielded by aluminum tubes

Mohamed H. Dadoura^{1✉}, Ahmed Ismail Zaky Farahat², M. R. Taha³ & Ramadan N. Elshaer^{4✉}

Aluminum closed cell foam blocks are created with a volume of 1 inch³ which consist of aluminum foam parts shielded with part of aluminum tube and in some types reinforced with inner aluminum tubes. Blocks have been made to overcome some existing problems in metallic foam used to protect some applications parts from impacts as a sacrificial part. Metallic foam has three main categories sandwich panels, filled tubes and corrugated sheets. Quasi-static compression tests have been applied on 12 blocks with different shapes and compared with pure aluminum foam blocks as a reference. Results display the enhancement of mechanical properties of blocks like yield strength (S_y), crushing strength (S_c) and densification strength (S_d), compression at strain 70%, as well as absorbed energy (area of compression under the curve). The highest value for yield strength (5.87 MPa) was registered for Finger phalanxes cube block (FP—0.1 Sq.). While the highest value for densification strength (21.7 MPa) was registered for spine cylinder block (SV8—0.17 C25). The registered results for samples apparent the highest value for energy dissipation density (E_{dd}) is 40.52 J/in³ (91% enhancement) and crushing strength (8.61 MPa) was registered for Finger phalanx cylinder block (FP—0.17 C25). The lowest value for E_{dd} is 14.16 J/in³ (less than pure aluminum foam block value by 33%), $S_y = 0.42$ MPa, $S_c = 3.21$ MPa, and $S_d = 4.46$ MPa, registered for thin wall Ear canal cylinder block (EC8—0.075 C26.5). Best mechanical properties had been achieved for Finger phalanx cylinder block (FP—0.17 C25) and spine cylinder block (SV8—0.17 C25).

Aluminum (Al) foam was fabricated in the middle of the last century. It was used in many applications such as supporting some parts of cars and containers to absorb shocks and enhance the isolation of sound and heat. Al closed cell foam (ACCF) is considered an expendable material in applications where it works as a sacrificial part that absorbs energy to protect parts or machines from hard impacts. Despite there are several forms of it like sandwich panels, filled tubes, and corrugated sheets used in industrial applications, it still faces some challenges like high cost of production and the high cost of casting parts or filling tubes, after impacts defective parts or sheets are replaced by new one as usual if possible, where it can't be repaired which lead to the high cost of maintenance (i.e. Al foam parts when expose to impact bending stress it will be deformed and must be replaced totally). Al foam was used in a lot of applications from lightweight building walls and roofs (i.e., steel Al sandwich (SAS) and Al foam sandwich panels (AFS)) to crashworthiness in automobiles.

Figure 1 shows some applications for aluminum foam parts that are used in shielding automotive frames against impacts also, trains used foam in crumple zones where it has lightweight and ability to absorb high energy during impact. The most important applications of metallic foam are crash boxes which shield the front bumper of cars. The most common type is a cylindrical canister or polygonal canister filled with foam as appears in Fig. 1e. Many shapes and models have been made for this box with different techniques like adding two parts of foam with different cell sizes where foam with big size face impact then put the one with less size (Fig. 1f) to improve energy absorption through gradual the shock absorption. The challenge still exists is the cost of maintaining the impacted parts plus the high cost of making this shield with controlled limited dimensions.

The solid foam material is classified into (1) Natural materials like human and animal bones, cancellous bone, cuttlefish bone and coral. (2) artificial materials like Steel foam, Aluminum foam and some cellular polymers⁴. ACCFBs have been invented from traditional components to overcome the cost of fabrication foam

¹Helwan University, Cairo, Egypt. ²Central Metallurgical Research and Development Institute, Cairo, Egypt. ³Faculty of Engineering, Cairo University, Cairo, Egypt. ⁴Tabbin Institute for Metallurgical Studies, Cairo, Egypt. ✉email: eng.muhamed.h.d@gmail.com; ramadan_elshaer@yahoo.com

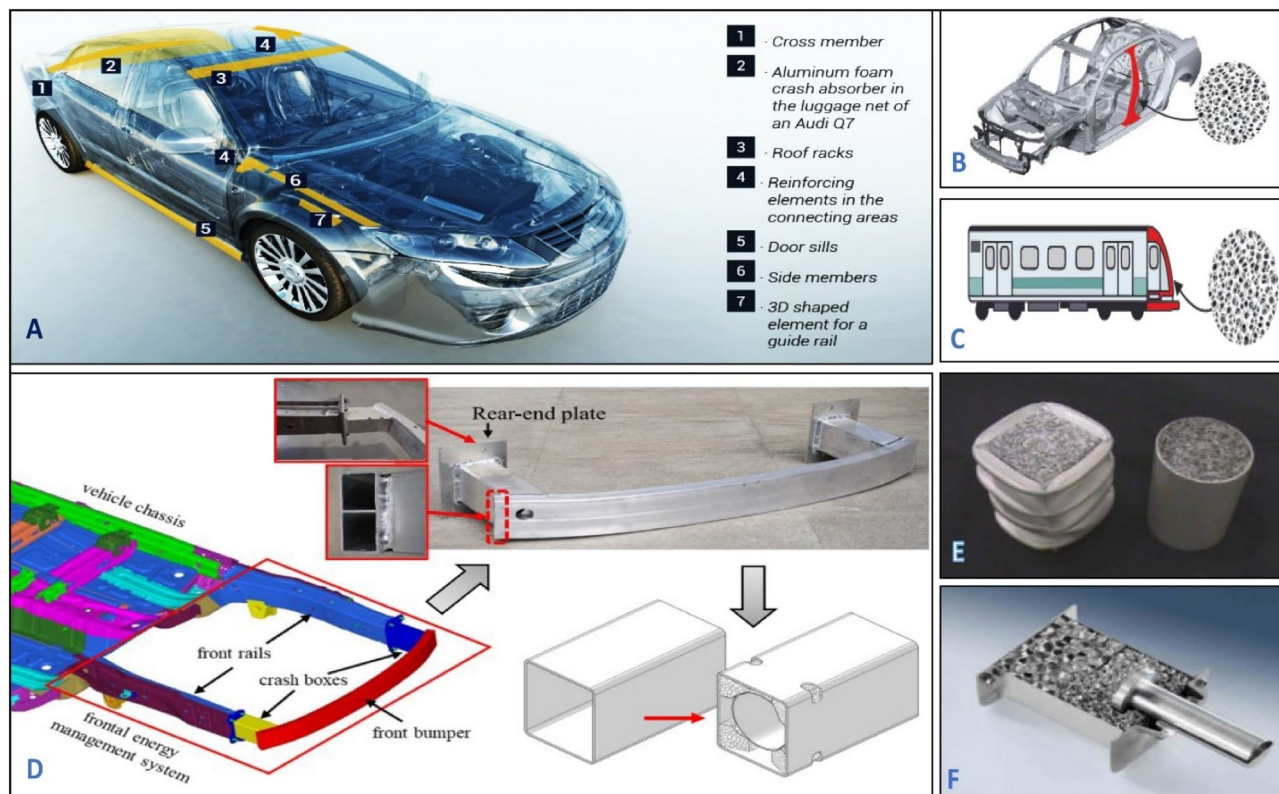


Figure 1. Some applications used of Al foam (a) Shielded parts by foam in automotive¹, (b) Shielding car frame by foam, (c) Crumble zone shielded by foam in trains, (d) Crash boxes of car front bumper², (e) Cylindrical and polygonal crash boxes, and (f) Crash boxes with two foam parts with different cell sizes³.

for nonuniform shapes and the cost of maintenance for uniform shapes. Bones are the optimum design created by great creators which combined compact bone and spongy bones. It can bear stresses and absorb high energy during motion the awesome thing is accurate distribution for dimensions of bones which are considered as a group of blocks that have limited sizes and this makes every part able to apply its function in easy way.

Three categories have been simulated by aluminum foam with different shapes and dimensions distribution for the same volume (1) Finger phalanx blocks where in real cases when collecting finger phalanx in punch position bones will be able to increase its energy absorption by 4 to 5 times. (2) Spine blocks which able to save expensive wire or cable and can control bending angle by putting flexible spacers between spine bones which simulate discs in real. Spine shields spinal cord from impacts despite its flexible motion in body. (3) Ear canal blocks are able to shield expensive cables too like spine blocks but with equal conditions surrounding their surface. Figure 2 shows the finger phalanx bone formation, lumbar spine photo and cross section, ear canal placed in cranium and cross section for it and femur bone cross section. From the mechanical point of view the muscles, fats, and skin are considered damping materials where it helps in reducing the impact of stress on bones.

Problems could be concluded in high cost of foam production, maintenance and high thermal isolation which is harmful to some applications. So, Blocks are made to simulate some bones ideas in human skeleton like bones of finger phalanx, spine (vertebrae) and ear canal bones in skull. The idea of this research came from meditation on skeleton bones where bones are created from calcium foam (spongy bone) shielded with a hard calcium layer (compact bone) and sometimes bones are reinforced with compact bones inside their part according to function and applied stresses which created to bear it like ear canal bones and spine bones.

Experimental work

The advantages of Al foam are high energy absorption through plastic deformation excellent vibration dampening, heat and sound isolation for a density bigger than 400 kg/m^3 and it can be recycled. The advantages of ACCFBs are: availability of tubes in traditional market by different, materials, sizes and thicknesses also, foam can be chosen according to its type, density and cell size. ACCFBs consist of aluminum foam which simulates spongy bone, and small parts of Al tubes to simulate compact bone shields. Al tubes have rectangular and circular shapes for outer shields and inner reinforcement circular tubes with small diameters tubes of 8 and 10 mm.

Aluminum Foam is a composite material defined as a special case of porous metals where a solid foam originates from a liquid foam in which gas bubbles are finely dispersed in a liquid with semi-equal sizes. Porous metal's relative density (P_{rel}) should not be greater than 70%. Metallic foam in common can arrive at 30%⁸. Note that a higher density of the foam means stiffness increases and energy absorption ability will be decreased.

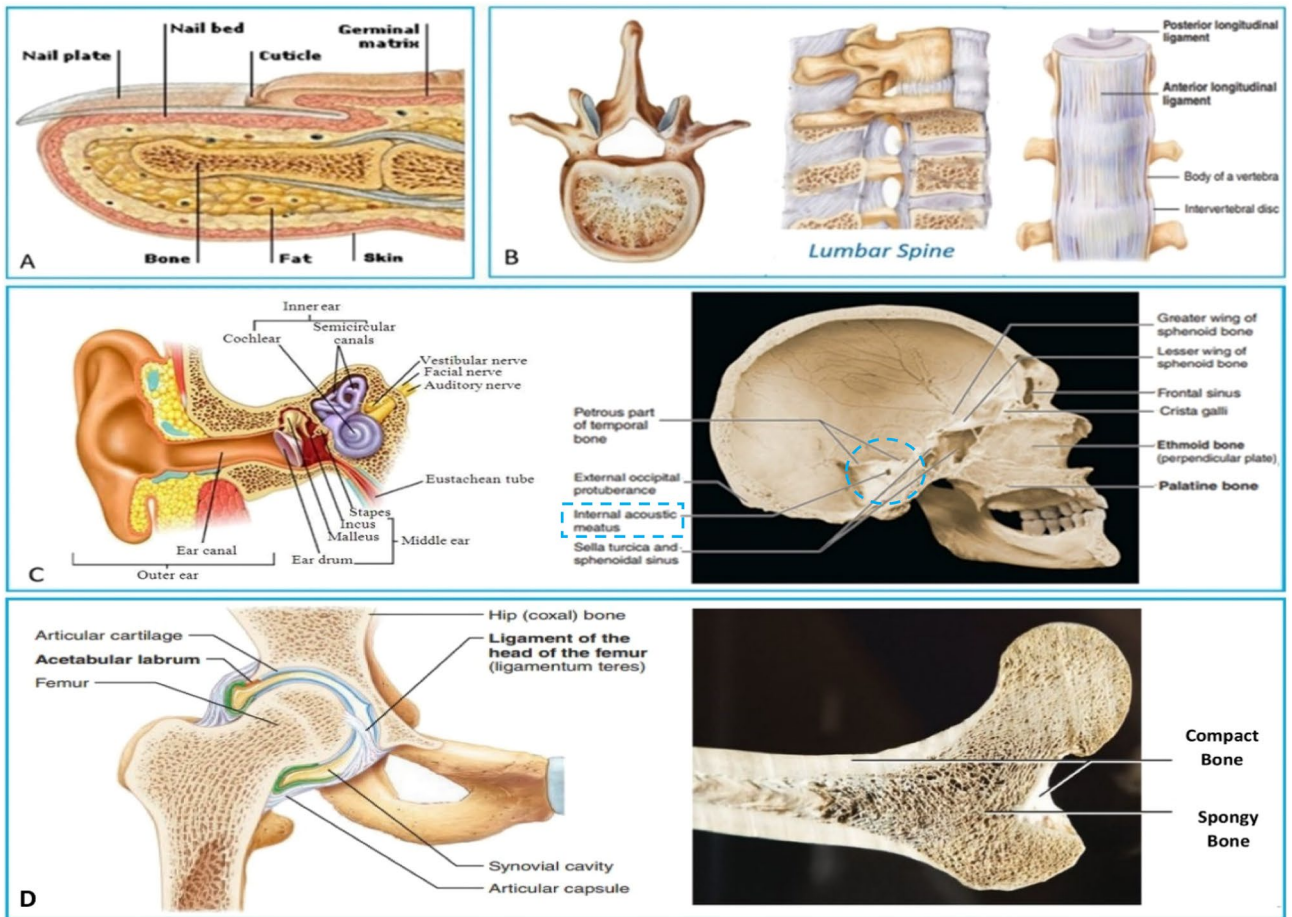


Figure 2. (a) Hand anatomy, phalanx bone formation⁵, (b) Spine (Lumbar Vertebrae)⁶, (c) Ear canal structure and components^{6,7}, and (d) Femur bone front section⁷.

$$\rho_{rel} = \rho_{foam} / \rho_{base\ metal} \quad (1)$$

Foam is defined as an amorphous alloy according to its crystal structure where the atoms have no opportunity to form a crystalline lattice and solidify in a disordered manner due to foam bubble formation. Foam cells consist of cell walls, Plateau borders, and nodes (see Fig. 3). *Cell wall*: separates two gas bubbles over a length of about the bubble diameter and shows a curvature that is much smaller than the mean curvature of the two bubbles. Generally, the mean cell wall thickness is much smaller than the bubble diameter. *Plateau borders*: defined as the intersections of the walls. *Nodes*: are junctions of at least the four Plateau borders, nodes formed when plateau borders are disordered and form a network⁹.

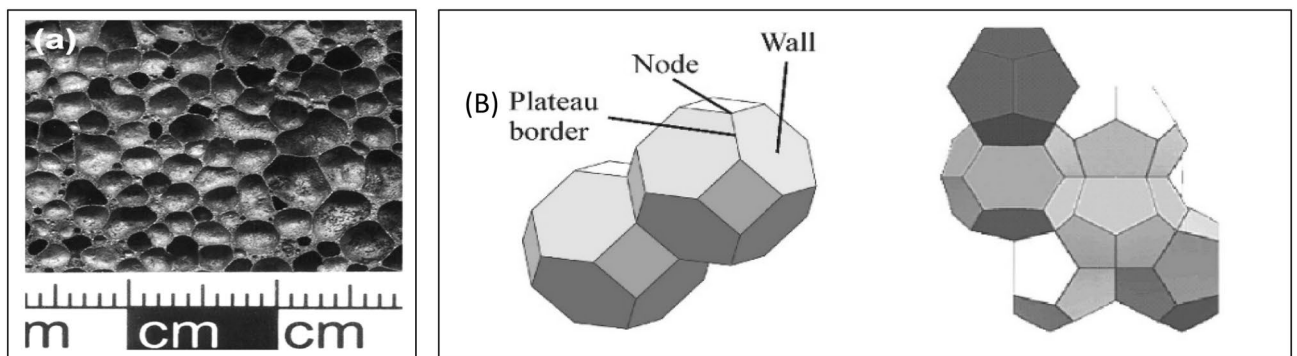


Figure 3. (a) Closed-cell Al foam produced by the precursor route with TiH₂¹⁰ and (b) Component of foam cells.

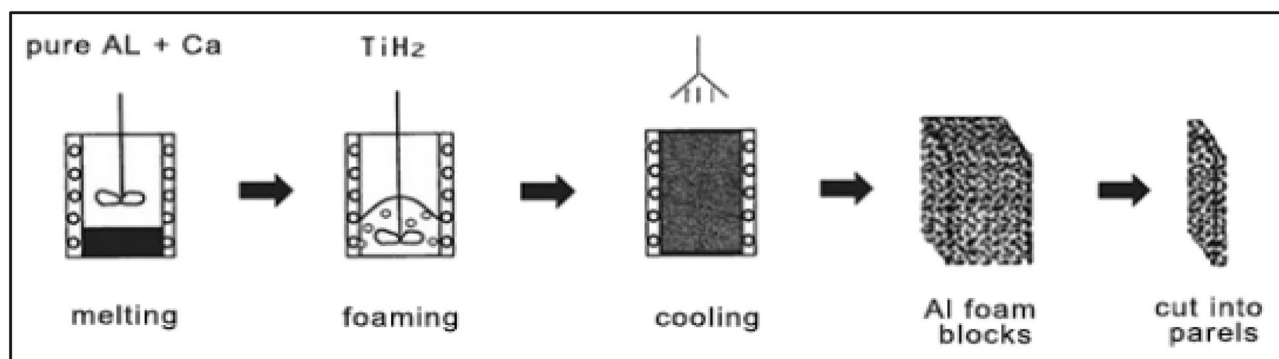


Figure 4. Closed-cell Al foam manufacturing procedures.

The manufacture data and properties of Al foam used in ACCFBs which conclude type, chemical composition (Base material, reinforcement material, precursor), cell size, density, and compressive strength at 70% strain are listed in Table 1. To ensure that all samples have equal properties all blocks have been cut from one sheet of Al foam with a density of 400 kg/m³ and wall thickness in a range of 0.2 to 0.23 mm. Figure 4 shows the procedures for producing Al foam by manufacture: Smelting pure aluminum matrix at 680 °C then adding thickening material (1.5 wt% Ca) and mixing it with aluminum melt after that transfer smelting mixture from smelting furnace to foaming furnace for viscosification and foaming process by add foaming agent (1.6 wt% TiH₂) and mix it with rotating impeller. Melt decomposes under the influence of heat and it releases hydrogen gas. As a result, the foam expands and fills up the mold within 15 to 20 min. After mold arrives to specified cell size cooling process will start with air or water after that slab will be ready to saw according to the required dimensions^{11,12}.

Aluminum tube are available in the traditional market dimensions are listed in Table 2. The chemical composition and designation are listed in Table 3 which agreed with standard DIN EN 755-2—AS/NZS 1866¹³. Figure 5 shows the photo of Al tubes that have been used.

ACCFBs fabricated by machining tools (i.e. Saw, files, drill and rubber hammer) for Al foam and tubes parts the blocks have been assembled according to their designed dimensions and adhesive epoxy metal has been used at the ends of blocks to confirm that parts assembled well together. Actually, there are two methods to make blocks as appears in Fig. 6 the first method is machining both Al foam and tube parts and then assembling

Al foam manufactured and supplied by: CHALCO ALUMINUM FABRICATION - China		
Type of foam: Alporas (Closed cell)	Cell size: 4 mm	Wall thickness: 0.22 mm
Chemical composition		
Base material: Al 1050	Void percent: 86%	Relative Density: 14%
Precursor: TiH ₂		
Thickening agent: Ca	Strain: 70%	Density: 0.378 g/cm ³
Compressive strength: 8.4 MPa		

Table 1. Properties of aluminum foam.

Tube cross-section	Square	Rectangular	Circular				
Dimension (mm)	25×25	20×40	Φ 26.5	Φ 25	Φ 30	Φ 8	Φ 10
Thickness (mm)	1	1	0.75	1.7	1.7	1.2	1.25

Table 2. Aluminum tubes dimensions and designation.

Chemical composition wt%									
Fe	Si	Mn	Cr	Ti	Cu	Mg	Zn	Others	Al
0.196	0.383	0.015	0.020	0.018	0.010	0.425	<0.001	0.023	98.91
Material designation: DIN EN 755-2—AS/NZS 1866						Grade: Al 6060			

Table 3. Chemical composition analysis and designation of Aluminum tubes material.



Figure 5. Aluminum 6060 hollow tubes.

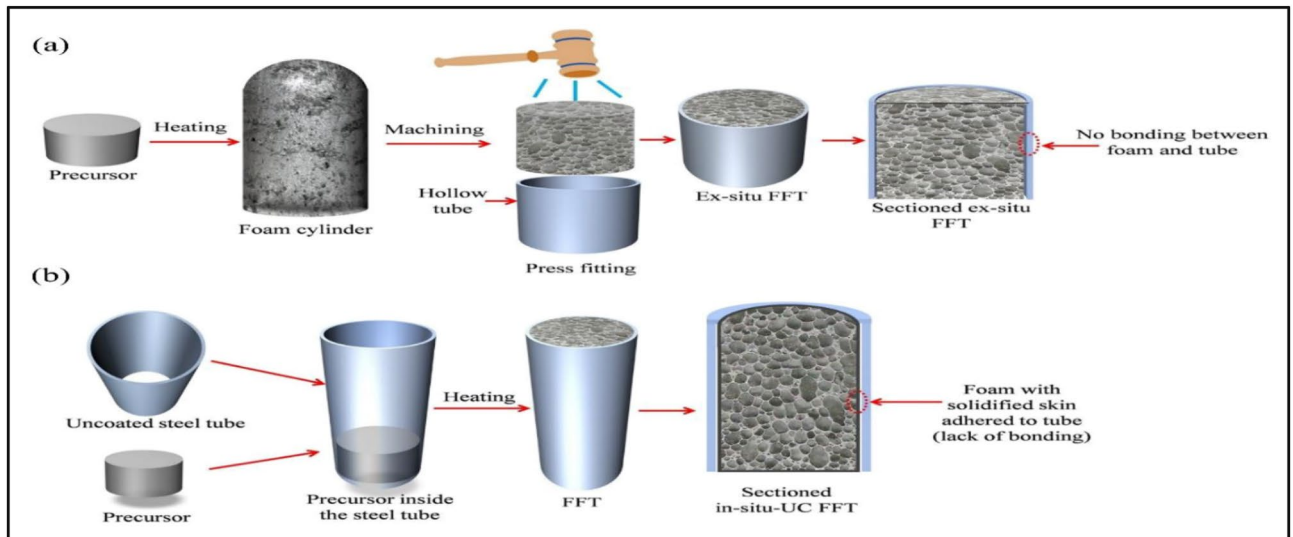


Figure 6. Schematic illustration of the foam filled tube fabrication of (a) external situ and (b) internal situ foam filling.



Figure 7. Samples of Aluminium foam Blocks with limited sizes.

them by soft hammering¹⁴. The second method is building up foam inside tubes profile by mixing first powder metallurgical matrix material, foaming agent (TiH₂ or ZrH) and additives (Mg, Si, ... etc.) then making cold compacting and then hot extruding at about 400–480 °C. The foaming agent thus becomes uniformly distributed and gas-tightly embedded in the metal matrix. The extrusion process is useful in helping to break up the oxide films on the surface of the metal powders, which facilitates consolidation. The product may be considered as a precursor material, itself not far from full density but readily convertible to foam. This conversion is affected by simply heating the precursor to a temperature at which the alloy is liquid. The foaming agent evolves gas, thus creating a foam that is stabilized by very fine oxide particles uniformly distributed throughout the precursor after extrusion. After melting and foaming, the foamed panel is rapidly cooled to prevent collapse of the foamed structure¹⁵. Although the second method will be producing more strengthen blocks and be cheaper in total cost of fabrication than the first method it is suitable and more reliable for mass production and accurate parts like prosthetics. The first method is easy, general, and gives variety in use where any type of foam can be selected with the required properties¹⁶, and selection of tubes with different dimensions is easy to be made by common man with limited block numbers according to needs.

ACCFBs twenty samples have been made as appears in Fig. 7. Twelve of them have been selected for applying lateral compression (quasi-static) test and comparing results with pure foam block to specify the enhancement values. The samples simulate four categories of bones: pure foam block, finger phalanxes, spine, and ear canal

Block type	Block shape	Sample code	Mass (gm)	Vol. at 1 inch ³ (cm ³)	Dimensions (mm)							Geometrical properties		
					X	Y _a	Y _v	Z	T	∅ _{in}	t	A (mm ²)	Z _p (mm ³)	I (mm ⁴)
Pure Foam		Pure—Sq	5.91	15.625	25	25	25	25	1	–	–	625	2604.2	32,552
Finger Phalanx blocks		FP—0.1 Sq	11.48	15.625	25	25	25	25	1	–	–	625	2604.2	32,552
		FP—0.1 Pa	11.43	16	40	16	20	20	1	–	–	800	2666.7	26,666
		FP—0.17 C25	15.18	15.707	∅ = 25	25	32	–	1.7	–	–	490.87	1534	19,175
		FP—0.075 C26.5	9.78	15.443	∅ = 26.5	24	28	–	0.75	–	–	551.55	1827	24,208
		FP—0.17 C30	13.60	15.550	∅ = 30	21	22	–	1.7	–	–	706.86	2650.7	39,761
Spine blocks		SV8—0.1 Sq	12.73	14.987	25	25	25	25	1	8	1.2	600.37	2540	31,119
		SV8—0.1 Pa	14	14.724	40	16	20	20	1	8	1.2	750.74	2617.5	25,339
		SV8—0.17 C25	16.80	14.890	∅ = 25	25	32	–	1.7	8	1.2	466.24	1479.9	17,939
		SV8—0.17 C30	14.70	14.988	∅ = 30	21	22	–	1.7	8	1.2	682.23	2669.9	42,245
Ear canal blocks		EC8—0.1 Sq	12.73	14.987	25	25	25	25	1	8	1.2	600.37	2466.2	32,504
		EC8—0.075 C26.5	11.18	14.753	∅ = 26.5	24	28	–	0.75	8	1.2	526.92	1689.1	24,160
		EC8—0.17 C30	14.70	14.988	∅ = 30	21	22	–	1.7	8	1.2	682.23	2512.8	39,712

Table 4. Pure foam, Finger phalanx, Spine and Ear canal blocks dimensions and codes. where X = Block length, Y_a = Block length for compression test, Y_v = Block length for volume 1 inch³, Z = Block height, ∅ = Circular block diameter, T = Shell tube thickness, ∅_{in} = Inner tube diameter, t = Inner tube thickness, A = Cross section area, I = Moment of area^{17,18}, Z_p = Section modulus¹⁹.

with different shapes (cube, parallelogram, and cylindrical) as appeared in Table 4. Quasi-static test has been applied and all curves data scaled to area cm².

Quasi-static compression tests have been applied on the universal testing machine (WDW-300 KN, China). The test velocity was 1 mm/min. Aluminum foam compression test was applied in accordance with standard “DIN 50,134:2008-10” of “Testing of metallic materials—Compression test of metallic cellular materials”²⁰. Where compressive strain (ε) equals change in length / original length.

$$\varepsilon = \Delta h / h \quad (2)$$

All types of aluminum foam in quasi-static compression tests have plastic collapse regions at strain between 65 and 75%. It is depending on the relative density, cell size, and material composition of the foam matrix. So, all ACCFB categories have been tested at strain 70%. There are two types of energy absorption criteria, which are specific energy absorption capacity (Es) and volumetric energy absorption (E_{ad}). Es can be defined as the total absorbed energy per unit mass and it is a performance index used in measuring the capacity of a material to absorb energy from an impacting load. It is defined as the ratio of maximum energy that can be dissipated by a unit of foam mass (m_f) and Ea is described as the potential energy of absorption which is equal to the area under the “stress–strain curve”^{21,22}.

$$Es = Ea / m_f \quad (3)$$

The energy-absorption capacity may also be expressed in terms of the average foam crush strength, (S_c), which is defined in foam: stress at which continuous plastic collapse begins. So, over a range of foam deformation. Es can be calculated, using the stress–strain curves produced by tests, assuming uniform loading is achieved.

$$E_s = (S_c d) / \rho \quad (4)$$

$$E_a = (S_c V_c) \quad (5)$$

where V is foam block volume (cm^3), V_c is compressed volume of foam block (cm^3), d is Foam deformation, V_c/V (dimensionless), ρ is Density of foam, (gm/cm^3).

Static energy-dissipation density of foams (E_{dd}) is a useful index to measure aluminum foam's energy absorption properties. This is the maximum energy that a unit volume of foam can absorb due to impact⁸.

$$E_{dd} = E_s \rho \quad (6)$$

The volume of ACCFBs have been chosen to be 1 inch^3 to study the amount of energy absorption due to dimensions redistribution on different geometrical shapes. Some body may have sound say impact test applied on effected area as impact toughness of solid metals or composites (i.e. Charpy test or Izod tests) but in real energy absorption in foam materials defined generally by E_{dd} .

Actually, impact is defined in mechanics of material as dynamic bending stress due to its velocity. Blocks are affected by a lot of factors like distribution of applied force on block shape, stiffness and strength of tubes energy absorption of both tubes and foam. under quasi-static test the foam will be compressed while tubes will expose to bending stress. So, moment of area can't be the only parameter for measuring stress behavior on block but shape dimensions, stiffness, strength, flexural rigidity and the effect of position of reinforcement tube (inner tube) control energy abortion of ACCFBs. The minimum limit for blocks volume is $1/2$ to $2/3 \text{ inch}^3$ according to its shield thickness and stiffness where under this limit E_{dd} will be reduced by about 20% so if block size needed to be reduced to $1/2 \text{ inch}^3$ its preferable to use foam twice relative density at least to be between (24 to 30%). This will increase density of foam and mass of block and also increase heat insulation capacity of foam. So, the selection of block volume to 1 inch^3 is optimum.

The Aluminum foam compression curve below exposes the stress–strain curve regions as appears in Fig. 8. Plateau collapse region could be specified through (S_c) by applying Eq. (4) or (5). The compressive stress–strain curve of metal foam has three main regions respectively: linear elastic region, plastic region where plateau collapse at the end of it, and densification region where foam density increases due to the full destruction of foam cells. The first region (linear elastic zone) occurred at a small strain (2–3%). The second region (plastic deformation) continues till about 70% strain. The third region (densification) will continue until the solid state²³.

Results and discussion

Compression test of pure Al foam block (Pure—Sq.) at strain 70%. Figure 9 shows the engineering stress–strain curve of pure aluminum foam cube. It is clear that the yield strength is 0.71 MPa while, compressive strength is 8.4 MPa (at strain 70%) and crushing strength (S_c) is 4.53 MPa (at strain 64.6%) and energy absorption (area of compression under the curve) is $E_a = 1.36 \text{ J}$ and $E_{dd} = 21.25 \text{ J}/\text{inch}^3$.

Compression test of finger phalanx blocks at strain 70%. Figures 10, 11, 12, 13, and 14 show the engineering stress–strain curve of finger phalanx blocks. It seems clear that the yield strength of the foam cube shielded with square tube (block: FP—0.1 Sq.) is the highest value (5.87 MPa) due to the resistance of square shape to deform. while the lowest yield strength is (0.43 MPa) for foam cylinder shielded with circular tube

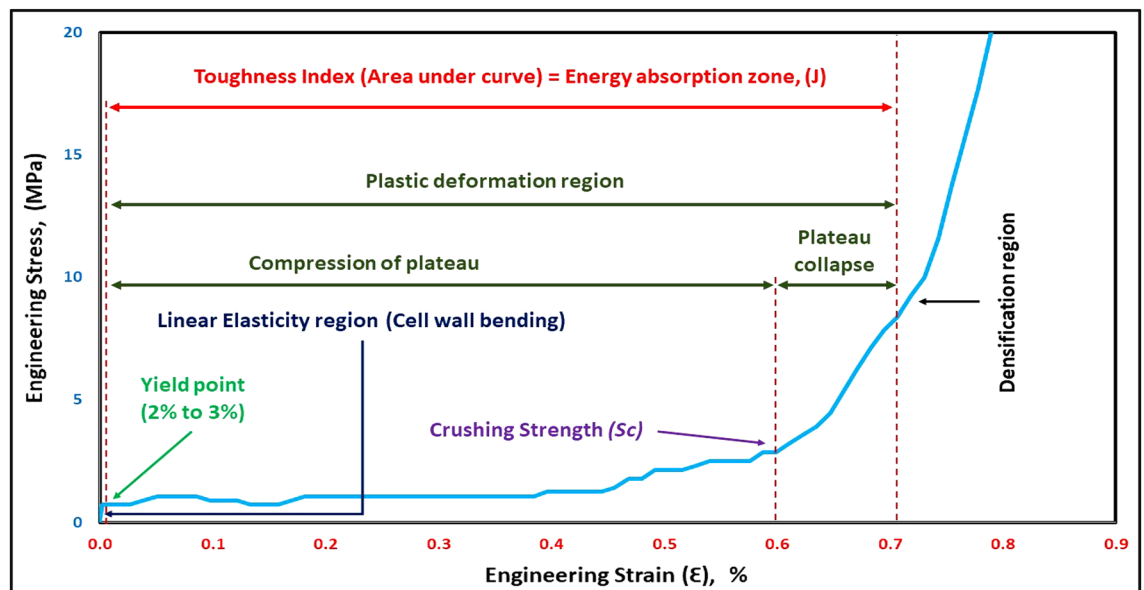


Figure 8. Stress–strain curve regions of Aluminium foam.

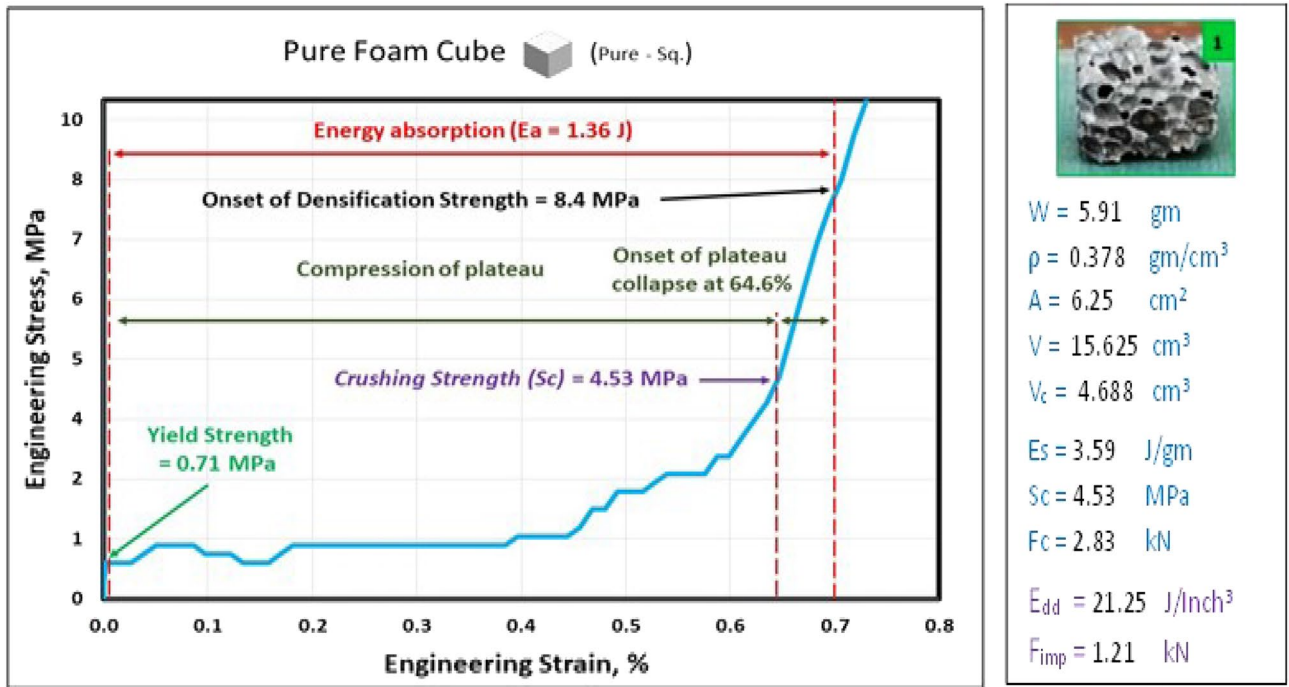


Figure 9. Stress–strain curve of pure aluminium foam block (Pure—Sq.).

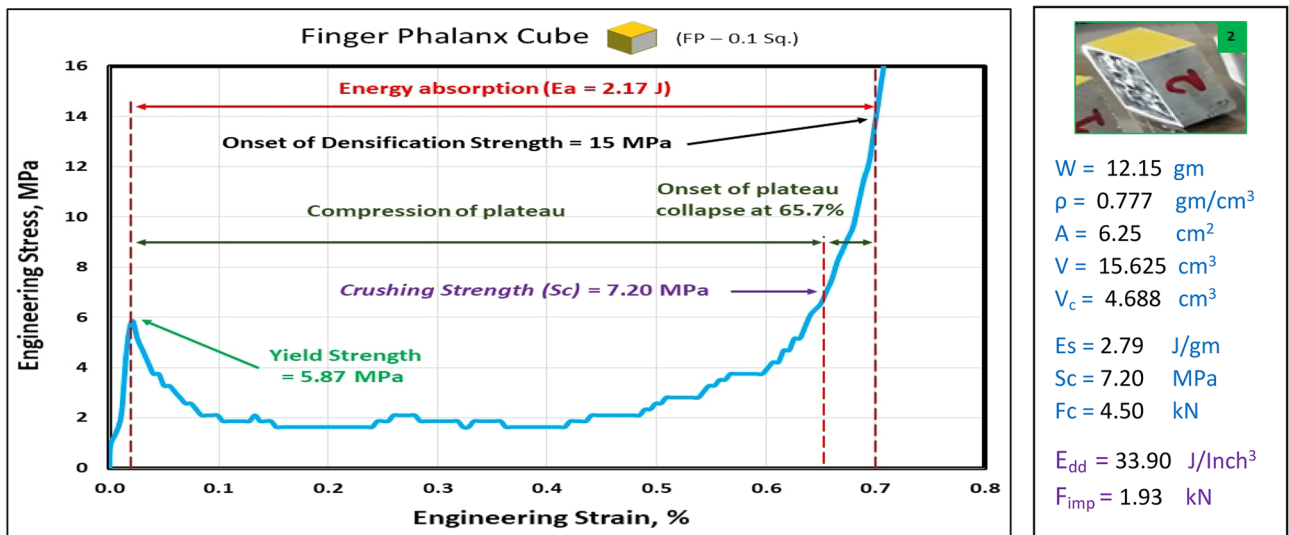
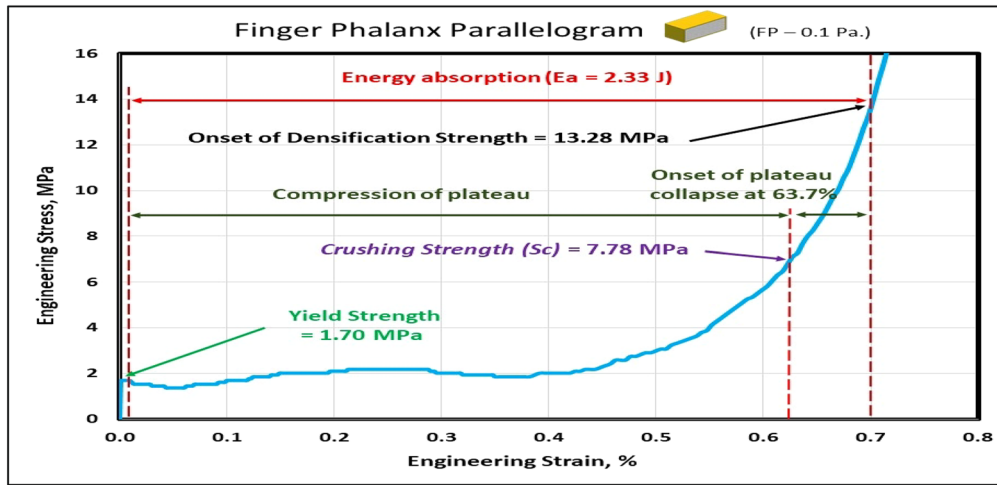


Figure 10. Stress–strain curve of finger phalanx cube block (FP—0.1 Sq.).

(block: FP—0.17 C30). This can be attributed to the large diameter of this shield. Also, it is apparent that the highest crushing strength (S_c) is (8.61 MPa) for foam cylinder shielded with circular tube (block: FP—0.17 C25) due to the small size of shield and the lowest (S_c) is (5.54 MPa) for foam cylinder shielded with circular tube (block: FP—0.17 C30) due to large size of shield where the resistance of deformation is reduced, and the collapse process of foam cells dominates that strength depends on cells shapes, wall thickness, size and distribution²⁴.

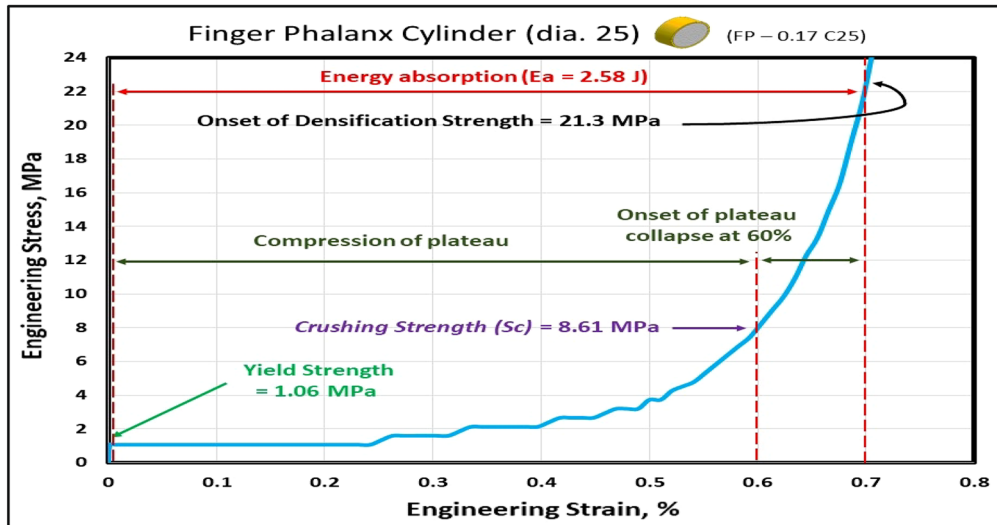
Figure 15 shows the summary of yield, crushing, and compressive strengths for all finger phalanx blocks by compared with pure aluminum foam blocks. Energy dissipation density (E_{dd}) has been calculated for all finger phalanx blocks at volume 1 inch³. Figure 16 shows energy dissipation density (E_{dd}) values for pure Al foam block and finger phalanx blocks with volume 1 inch³. Which have been rounded to the nearest whole numbers. This is defined easily as the toughness index for blocks where values have come from energy absorbed which is calculated by area under the curve and then multiplied by volume of the block.

Table 5 exposes the E_{dd} deformation length at strain 70% and energy absorption enhancement percent of blocks versus aluminum foam blocks in, where:



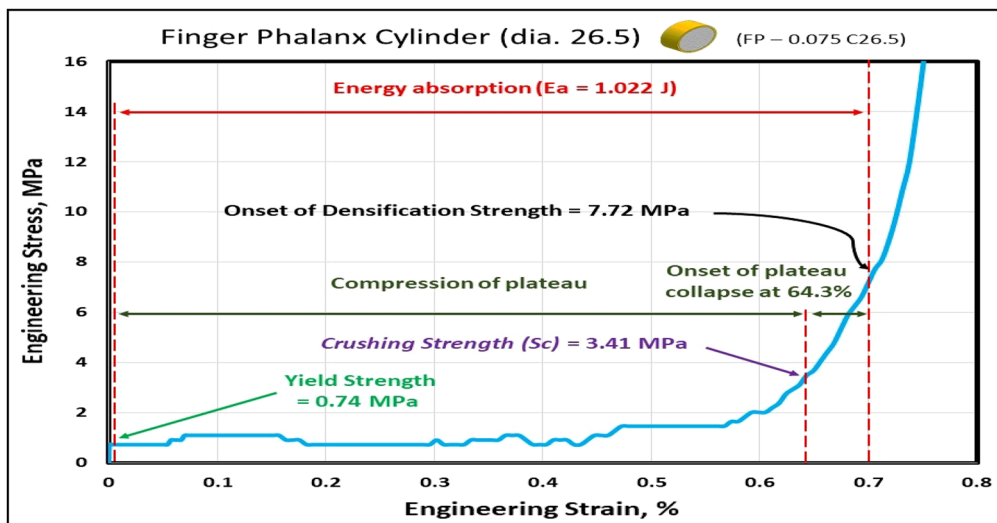
$W = 11.43 \text{ gm}$
 $\rho = 0.714 \text{ gm/cm}^3$
 $A = 6.40 \text{ cm}^2$
 $V = 12.800 \text{ cm}^3$
 $V_c = 3.840 \text{ cm}^3$
 $E_s = 3.27 \text{ J/gm}$
 $Sc = 7.78 \text{ MPa}$
 $F_c = 4.98 \text{ kN}$
 $E_{dd} = 37.28 \text{ J/Inch}^3$
 $F_{imp} = 2.66 \text{ kN}$

Figure 11. Stress–strain curve of finger phalanx parallelogram block (FP–0.1 Pa.).



$W = 15.03 \text{ gm}$
 $\rho = 0.956 \text{ gm/cm}^3$
 $A = 6.25 \text{ cm}^2$
 $V = 12.271 \text{ cm}^3$
 $V_c = 3.681 \text{ cm}^3$
 $E_s = 2.70 \text{ J/gm}$
 $Sc = 8.61 \text{ MPa}$
 $F_c = 5.38 \text{ kN}$
 $E_{dd} = 40.52 \text{ J/Inch}^3$
 $F_{imp} = 2.31 \text{ kN}$

Figure 12. Stress–strain curve of finger phalanx cylinder block (FP–0.17 C25).



$W = 9.95 \text{ gm}$
 $\rho = 0.644 \text{ gm/cm}^3$
 $A = 6.36 \text{ cm}^2$
 $V = 13.237 \text{ cm}^3$
 $V_c = 3.971 \text{ cm}^3$
 $E_s = 1.58 \text{ J/gm}$
 $Sc = 3.41 \text{ MPa}$
 $F_c = 2.17 \text{ kN}$
 $E_{dd} = 15.79 \text{ J/Inch}^3$
 $F_{imp} = 0.85 \text{ kN}$

Figure 13. Stress–strain curve of finger phalanx cylinder block (FP–0.075 C26.5).

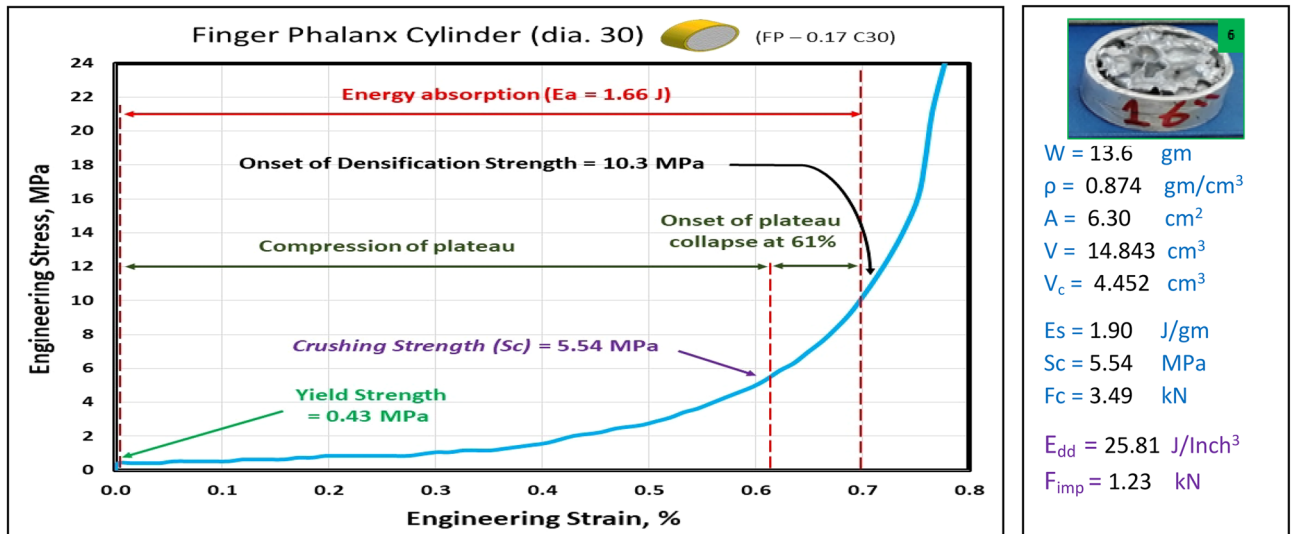


Figure 14. Stress–strain curve of finger phalanx cylinder block (FP–0.17 C30).

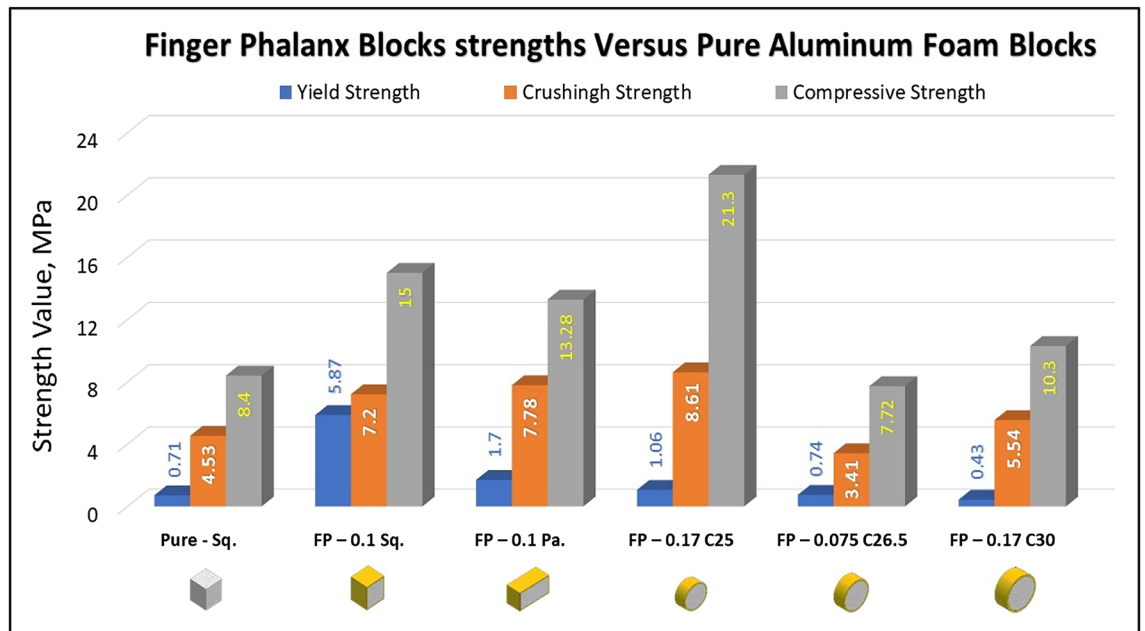


Figure 15. Finger phalanx blocks yield, crushing, and compressive strengths vs Al foam block.

$$\text{Enhancement percent} = (E_{dd}(\text{block}) - E_{dd}(\text{Al})) / E_{dd}(\text{Al}) \quad (7)$$

Enhancement percent calculations appear that the highest value for block FP-0.17 C25 by 91% and the lowest value for block FP-0.075 C26.5 is less than the pure Al block value by 26%. Results show that energy absorption (E_{dd}) is proportional to crushing strength (S_c) while densification strength (S_d) is proportional to density, shape, dimensions and thickness of shield tubes of blocks. Also, rectangular and circular shape tubes with larger sizes at the same thicknesses will be failure easier than shapes with smaller sizes which need high compression load for failure. Thin wall tubes are able to deform more easily than thick walls for tubes that have the same length^{25,26}. So, block FP-0.17 C25 has the highest S_c and the highest S_d while block FP-0.075 C26.5 has the lowest S_c and the lowest S_d .

Compression test of Spine (Vertebrae) blocks at strain 70%. Figures 17, 18, 19 and 20 show the engineering stress–strain curves of Spine (vertebrae) blocks. It seems clear that the yield strength of foam parallelogram shielded with rectangular tube and containing two offset inner tubes (block: SV8–0.1 Pa.) is the highest value (2.2 MPa) due to the resistance of rectangular shape to deform. While the lowest yield strength is (0.45 MPa) for foam cube shielded with square tube and containing one offset inner tube (block: SV8–0.1 Sq.). This can be attributed to the low resistance of the foam cube due to its little size and existence of the inner

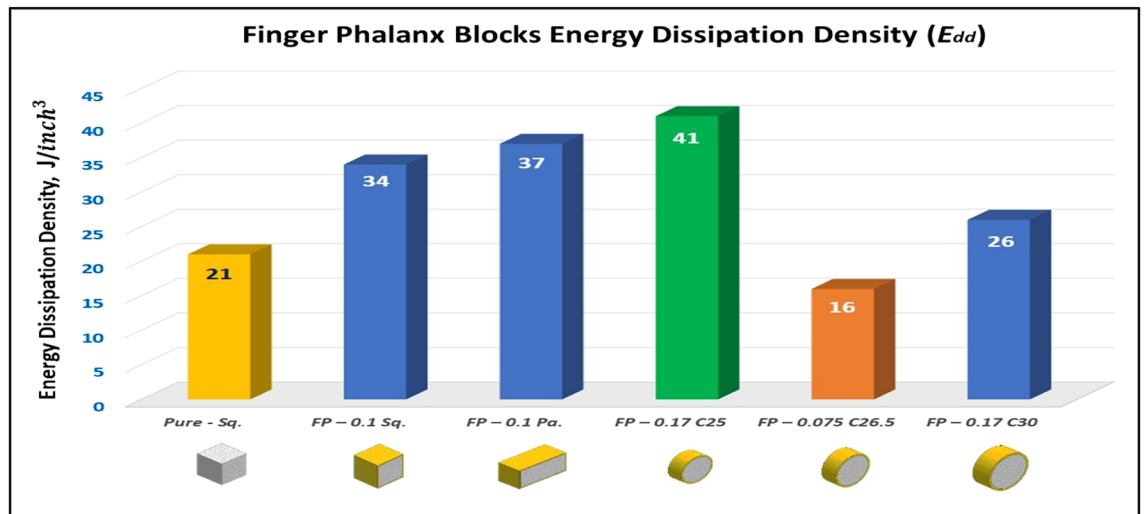


Figure 16. Al foam block and finger phalanx blocks toughness index (total absorbed energy per volume inch³).

Block code	Pure-Sq	FP-0.1Sq	FP-0.1 Pa	FP-0.17 C25	FP-0.075 C26.5	FP-0.17 C30
Deformation (mm)	17.5	17.5	14	17.5	18.55	21
E_{dd} (J/ Inch ³)	21.25	33.9	37.28	40.52	15.79	25.81
Enhancement (%)	Reference	60%	75%	91%	-26%	21%

Table 5. Blocks deformation length at strain 70% and energy absorption enhancement percent. Significant values are in [bold].

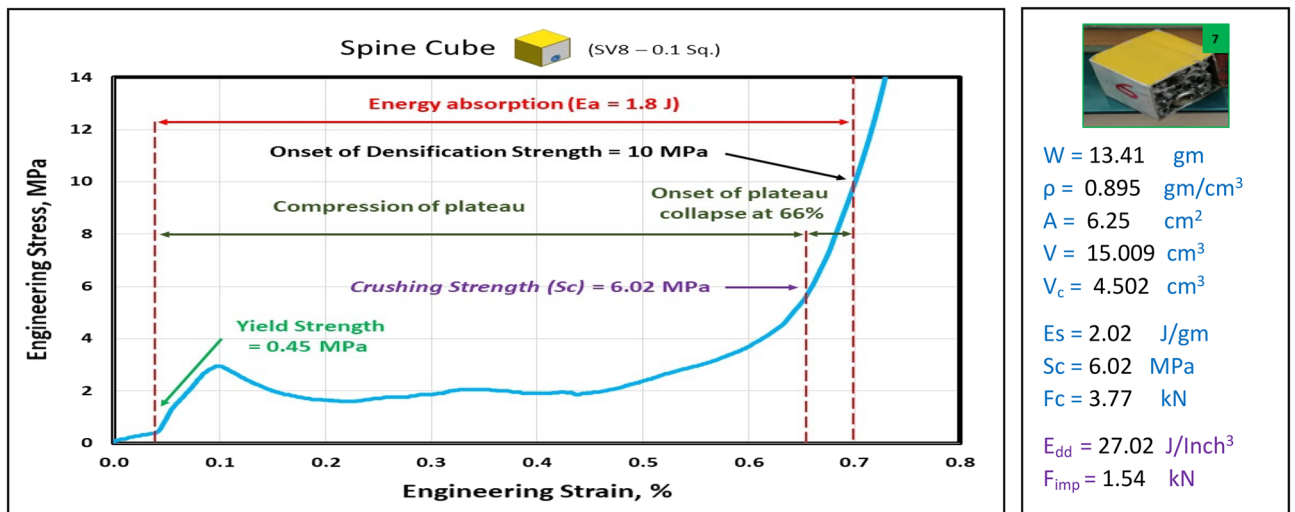
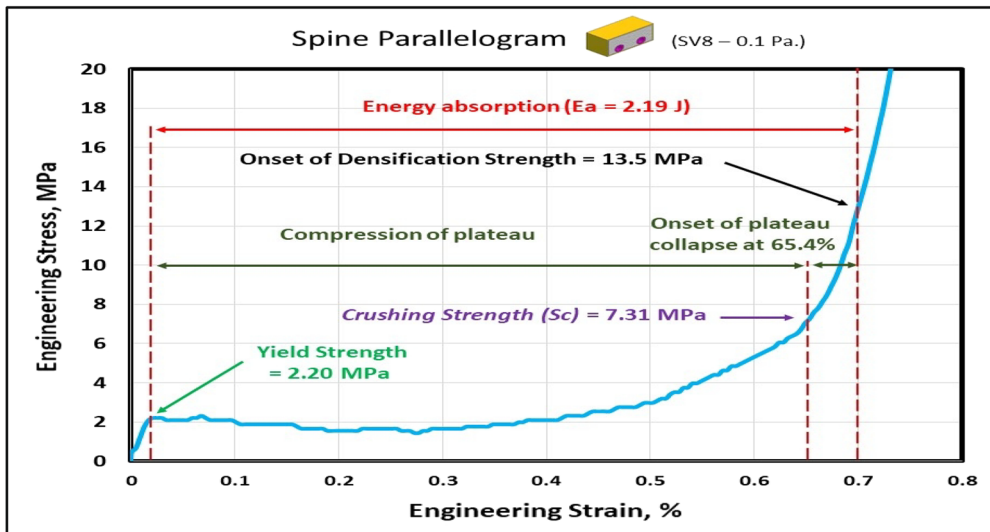


Figure 17. Stress–strain curve of spine cube block (SV8–0.1 Sq.).

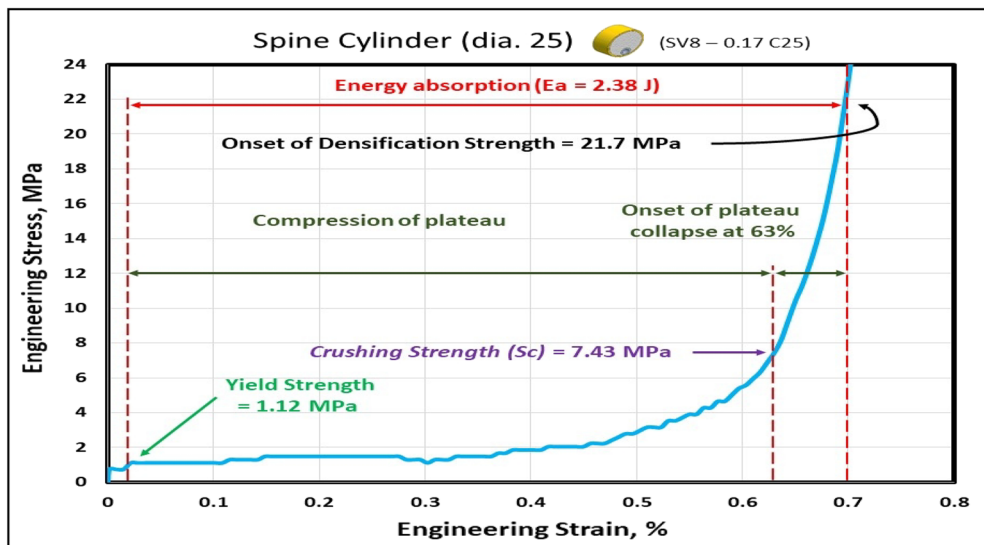
tube. Also, it is apparent that the highest crushing strength (S_c), compressive strength and energy absorption of 7.43 MPa, 21.7 MPa and 2.38 J, respectively is for foam cylinder shielded with circular tube (*block: SV8–0.17 C25*) due to the small size of shield and high resistance of inner tube where it compressed till about 20% strain. The lowest (S_c), compressive strength and energy absorption of 4.02 MPa, 9.47 MPa and 1.48 J, respectively is for foam cylinder shielded with circular tube (*block: SV8–0.17 C30*) due to the large size of shield where the resistance of deformation is reduced also at strain 70% inner tube will not be compressed. So, this large size will be suitable to guard expensive electrical wires.

Figure 21 displays the summary of yield, crushing, and compressive strengths for all spine blocks by compared with pure aluminum foam blocks. Energy dissipation density has been calculated for all Spine blocks at volume



W = 11.32 gm
$\rho = 0.961 \text{ gm/cm}^3$
A = 6.40 cm ²
V = 11.780 cm ³
V _c = 3.534 cm ³
E _s = 2.40 J/gm
Sc = 7.31 MPa
F _c = 4.68 kN
E _{dd} = 33.96 J/Inch ³
F _{imp} = 2.42 kN

Figure 18. Stress-strain curve of Spine Parallelogram block (SV8—0.1 Pa.).



W = 13.22 gm
$\rho = 1.134 \text{ gm/cm}^3$
A = 6.25 cm ²
V = 11.656 cm ³
V _c = 3.497 cm ³
E _s = 2.10 J/gm
Sc = 7.43 MPa
F _c = 4.64 kN
E _{dd} = 35.51 J/Inch ³
F _{imp} = 2.02 kN

Figure 19. Stress-strain curve of Spine Cylinder block (SV8—0.17 C25).

1 inch³. Figure 22 shows Energy dissipation density (E_{dd}) values for pure Al foam block and Spine blocks with volume 1 inch³ which have been rounded to the nearest whole numbers.

Table 6 exposes the E_{dd} with deformation length at strain 70% and enhancement percent of blocks vs aluminum foam block in energy absorption, by applying Eq. (7). Enhancement percent calculations appear that the highest enhancement value for block SV8-0.17 C25 by 67% and the lowest value for block SV8-0.17 C30 by 14%. Results show that E_{dd} is proportional to S_c while densification strength (S_d) is proportional to shape, dimensions and thickness of shield tubes of blocks. So, block SV8-0.17 C25 has the highest S_c and the highest S_d too while block SV8-0.17 C30 has the lowest S_c and the lowest S_d due to its shield tube large diameter. Rectangular block SV8-0.1 Pa is the best choice because it has ability to pass two wires through its inner tubes as a saver against impact and has good energy absorption.

Compression test of Ear canal blocks at strain 70%. Figures 23, 24, and 25 show the engineering stress-strain curve of Ear canal blocks. It seems clear that the yield strength and crushing strength of foam cube shielded with square tubes and containing inner tubes in the center (block: EC8—0.1 Sq.) is the highest value of 1.16 and 5.3 MPa, respectively. While the lowest yield strength, crushing strength, compressive strength and energy absorption of 0.42 MPa, 3.21 MPa, 4.46 MPa, and 0.96 J, respectively for foam cylinder shielded with a thin-walled tube which contains inner tube in the center (block: EC8—0.075 C26.5). This can be attributed to the low resistance of thin wall tubes and the distribution of foam cells in a cylinder shape with 8 mm pore in the middle which reduces its resistance against compression.

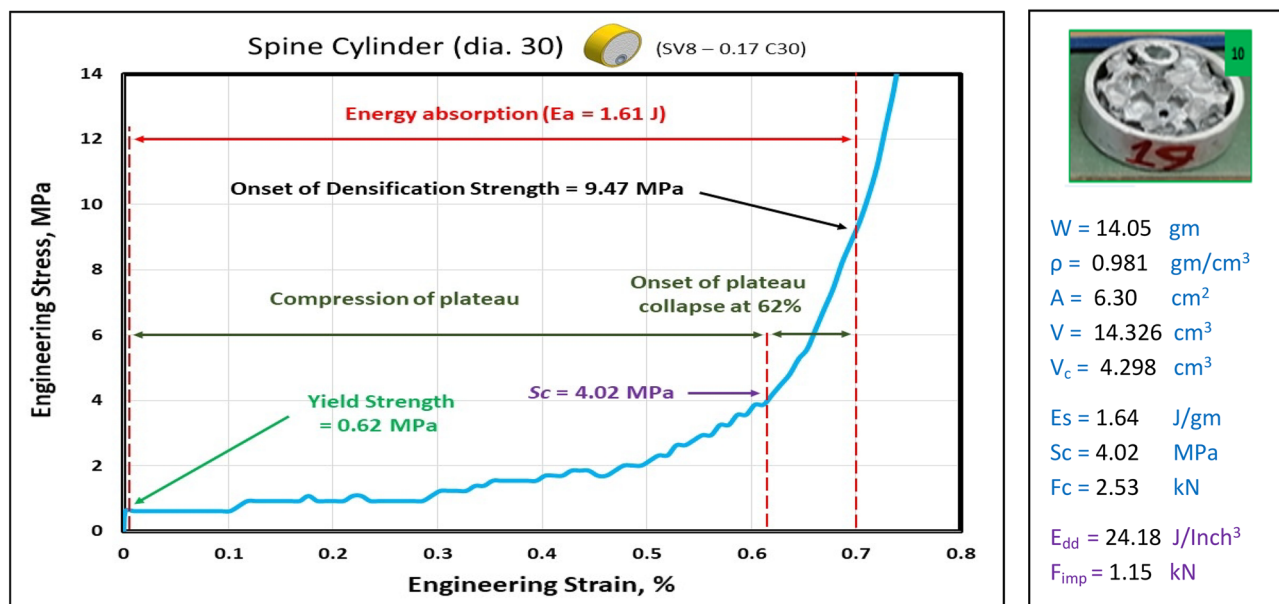


Figure 20. Stress–strain curve of Spine Cylinder block (SV8—0.17 C30).

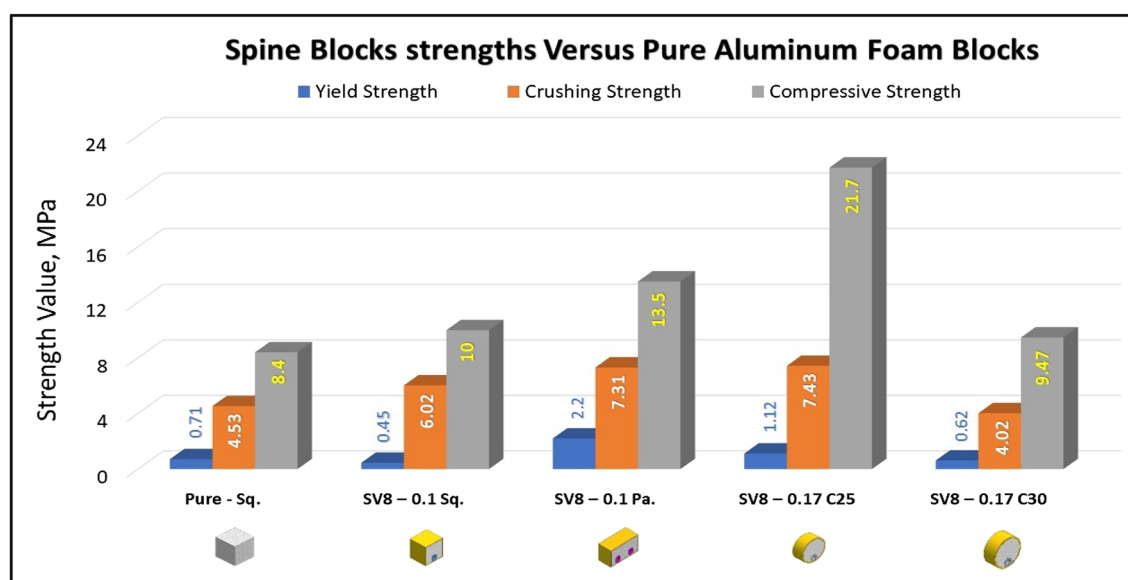


Figure 21. Spine blocks yield, crushing, and compressive strengths vs Al foam block.

Figure 26 shows the summary of yield, crushing, and compressive strengths for all Ear canal blocks by compared with pure aluminum foam blocks. Figure 27 displays the energy dissipation density (E_{dd}) values for pure Al foam block and Ear canal blocks with volume 1 inch^3 which have been rounded to the nearest whole numbers.

Table 7 exposes the E_{dd} with deformation length at strain 70% and enhancement percent of blocks vs aluminum foam block in energy absorption, by applying Eq. (7). Enhancement percent calculations appear that the highest enhancement value for block EC8-0.1Sq by 12% and the lowest value for block EC8-0.075 C26.5 is less than pure Al foam by 33%. Results show that E_{dd} is proportional to S_c while densification strength (S_d) is proportional to shape, dimensions and thickness of shield tubes of blocks. So, block EC8-0.1Sq has the highest S_c and the second arranged for highest S_d this refers to its square shape which resists compressive force by uniform projected area this means it needs to increase the load to deform despite it having a thin wall shield so its enhancement percent is relatively little. While block EC8-0.075 C26.5 has the lowest S_c and the lowest S_d due to its thin wall shield tube, relatively little length and circular shape.

Finally, a lot of parameters control the deformation mechanism of blocks under quasi-static test which affects energy absorption values like shield shape, thickness and type of material. Also, foam material, shape and dimensions distribution, cell size, foam density, and foam component and its composition. Simply all aforementioned samples can be concluded in one table to register the energy absorption properties needed for different

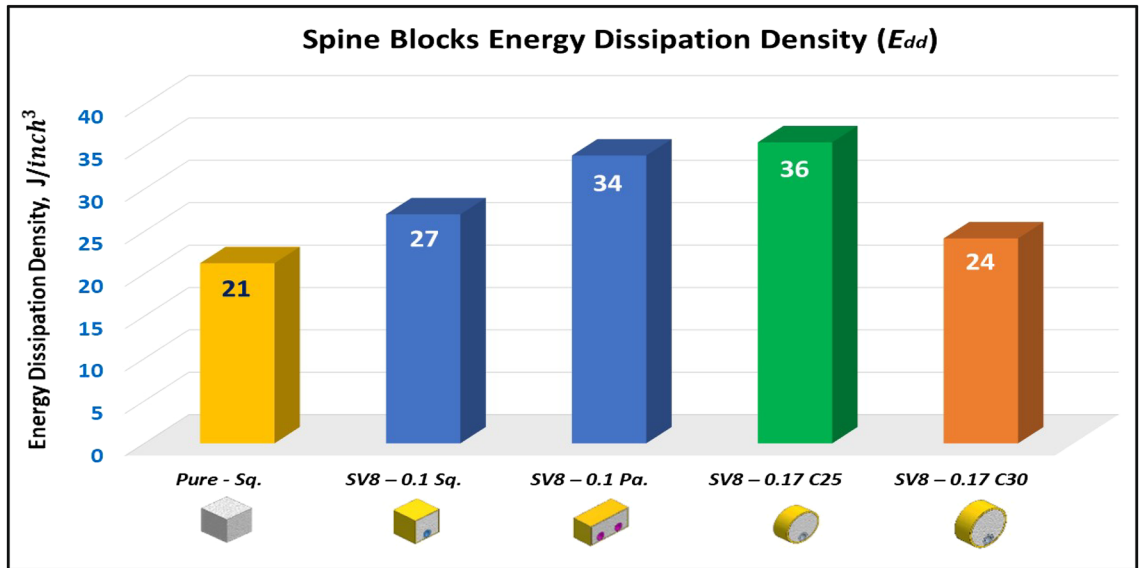
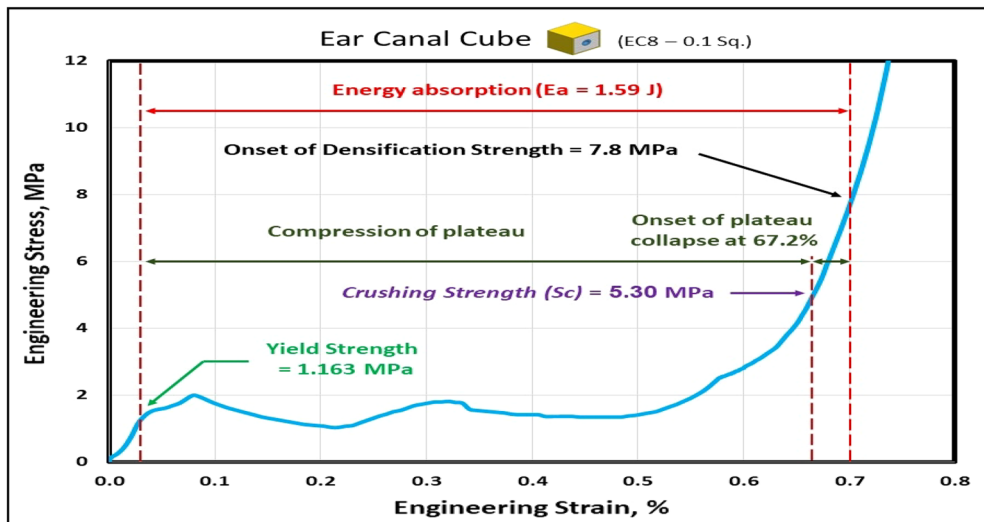


Figure 22. Al foam block and Spine blocks toughness index (total absorbed energy per volume $inch^3$).

Block code	Pure-Sq	SV8-0.1Sq	SV8-0.1 Pa	SV8-0.17 C25	SV8-0.17 C30
Deformation (mm)	17.5	17.5	14	17.5	21
E_{dd} ($J/Inch^3$)	21.25	27.02	33.96	35.51	24.18
Enhancement (%)	Reference	27%	60%	67%	14%

Table 6. Blocks deformation length at strain 70% and energy absorption enhancement percent. Significant values are in [bold].

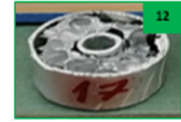
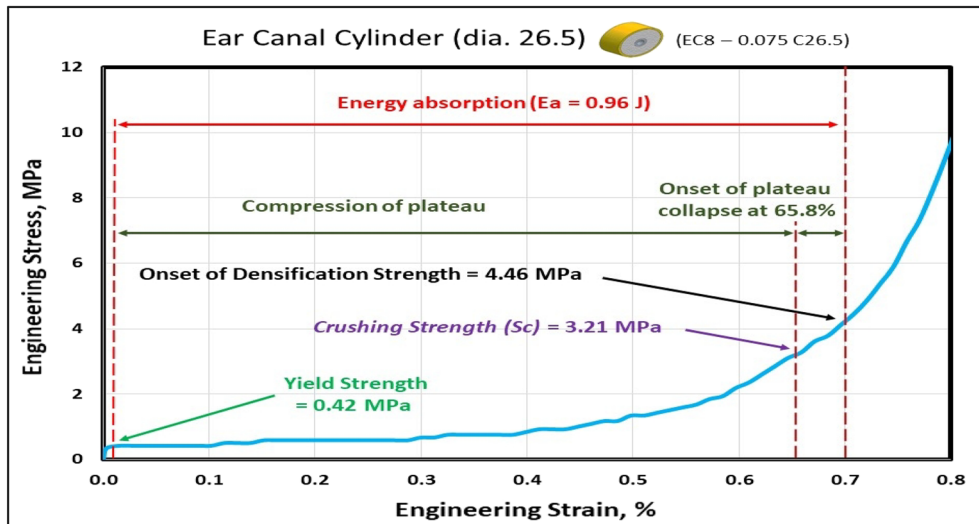


$W = 13.37 \text{ gm}$
 $\rho = 0.892 \text{ gm/cm}^3$
 $A = 4.90 \text{ cm}^2$
 $V = 14.987 \text{ cm}^3$
 $V_c = 4.496 \text{ cm}^3$
 $E_s = 2.30 \text{ J/gm}$
 $Sc = 5.30 \text{ MPa}$
 $F_c = 2.60 \text{ kN}$
 $E_{dd} = 23.86 \text{ J/Inch}^3$

Figure 23. Stress–strain curve of Ear Canal Cube block (EC8—0.1 Sq.).

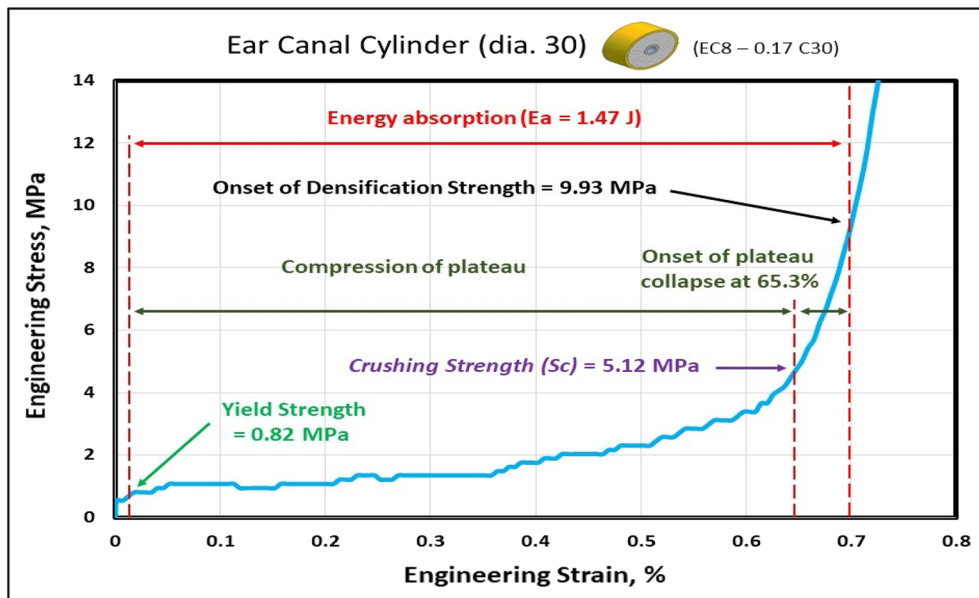
applications: energy absorption per volume 1 cubic inch (E_{dd}) at 70% Strain, and mass. Table 8 exposes properties of all blocks Deformation height at 70% strain, mass and energy absorption for each block.

Applications of ACCFBs. The main idea of ACCFBs is their flexibility in use and maintained easily where it is expendable parts can be collected in main three categories as Fig. 28 illustrate (1) Similar blocks pattern which consists of one type of block, (2) Multi blocks pattern consisting of different blocks types, (3) Combined blocks pattern consists of other components plus blocks like memory foam or silicon slices, rubber sleeves or metal canister. Actually, the expansion deformation factor should be considered where shield tubes deformed



$W = 10.06 \text{ gm}$
 $\rho = 0.796 \text{ gm/cm}^3$
 $A = 5.06 \text{ cm}^2$
 $V = 12.645 \text{ cm}^3$
 $V_c = 3.794 \text{ cm}^3$
 $E_s = 3.02 \text{ J/gm}$
 $Sc = 3.21 \text{ MPa}$
 $F_c = 1.63 \text{ kN}$
 $E_{dd} = 14.16 \text{ J/Inch}^3$
 $F_{imp} = 0.77 \text{ kN}$

Figure 24. Stress–strain curve of Ear Canal block (EC8—0.075 C26.5).



$W = 14.54 \text{ gm}$
 $\rho = 1.015 \text{ gm/cm}^3$
 $A = 5.16 \text{ cm}^2$
 $V = 14.326 \text{ cm}^3$
 $V_c = 4.113 \text{ cm}^3$
 $E_s = 1.512 \text{ J/gm}$
 $Sc = 5.12 \text{ MPa}$
 $F_c = 2.64 \text{ kN}$
 $E_{dd} = 22.06 \text{ J/Inch}^3$
 $F_{imp} = 1.05 \text{ kN}$

Figure 25. Stress–strain curve of Ear Canal block (EC8—0.17 C30).

and expand in width from 115 to 140% according to tube shape while, foam expands in all directions due to impact slightly by about 105%.

The total energy absorption of patterns can be estimated easily at strain 70% but two parameters should be specified besides energy pattern area and total mass. Example of the estimated of total energy absorption of Fig. 28b pattern by referring to Table 8 for calculations in Table 9.

The results of the total energy absorption of pattern are 813 J if all blocks have been replaced with one part of aluminium foam will reduce this area to half but actually, it will give total $E_{dd} = 756 \text{ J}$ and mass will reduce by about 35% than pattern. Seriously there are some challenges faced aluminium foam panels in applications where foam is good insulator for heat also, can't pass cables or wire through it and this gives limitation on use big parts of foam in vehicles especially for wide areas will need high amount of panels furthermore it valid to receive impacts in one plan this mean to cover two plans (i.e., XZ, YZ) or oblique impact will need two surfaces which will reflect on amount of needed foam. Also, foam as solid (not flexible) part not suit nonuniform shapes which lead to involved in expensive process like casting or accurate machining.

Figure 29 exposes the Evaluation of the collision types and its percentage²⁷. Figures 30, 31, 32 illustrates the developed types of combined patterns by research team where polyurethane memory foam (PU-220) with density 220 kg/m³ has been used as cushioning and energy absorber material. The peak cushioning efficiency properties at quasi static test were $E_{dd} = 0.13 \text{ J/cm}^3$ (2 J/in³), strain 57%, compressive stress was 0.44 MPa, 39% energy return²⁸. It is designed to suit most vehicles with considering the aforementioned challenges.

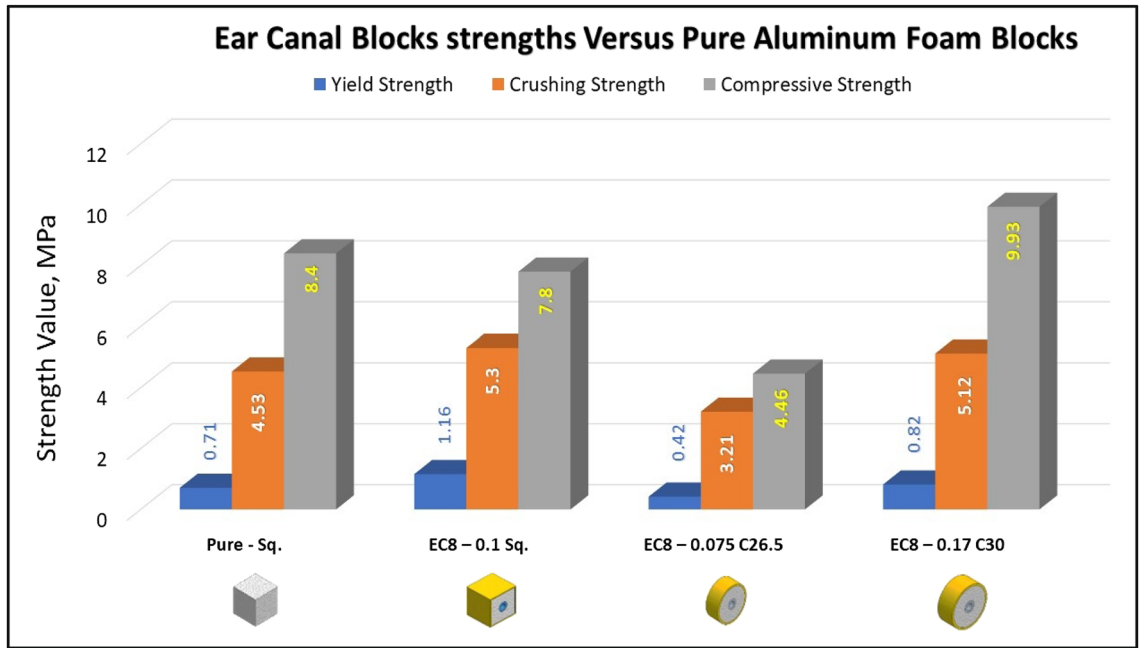


Figure 26. Ear canal blocks yield, crushing, and compressive strengths vs Al foam block.

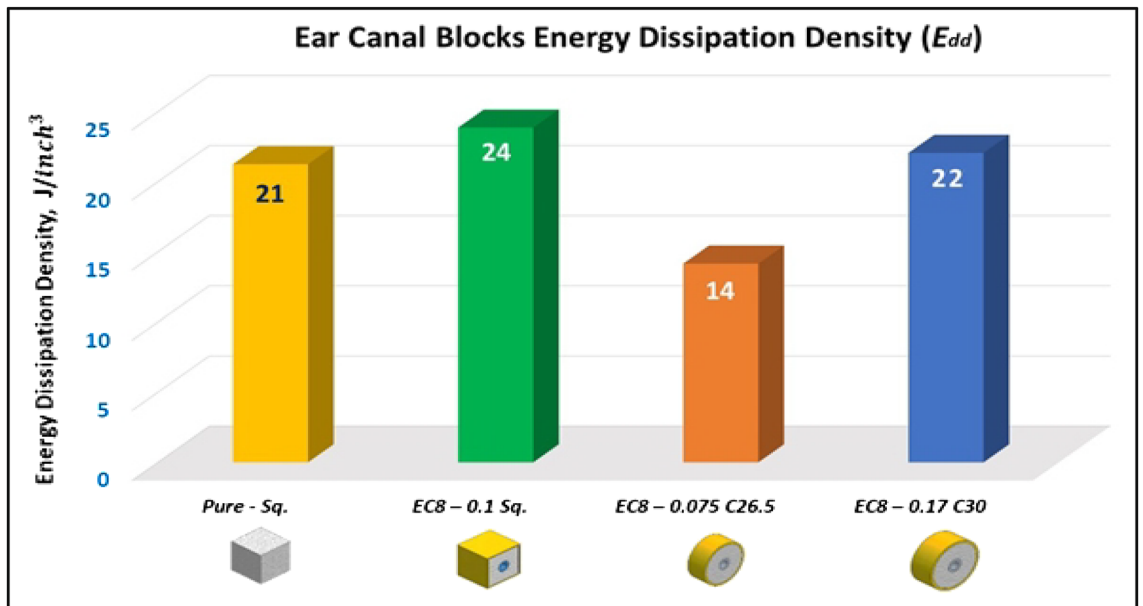


Figure 27. Al foam block and Ear canal blocks toughness index (total absorbed energy per volume inch³).

Block code	Pure-Sq	EC8-0.1Sq	EC8-0.075 C26.5	EC8-0.17 C30
Deformation (mm)	17.5	17.5	17.5	21
E_{dd} (J/ Inch³)	21.25	23.86	14.16	22.06
Enhancement (%)	Reference	12%	-33%	4%

Table 7. Blocks deformation length at strain 70% and energy absorption enhancement percent. Significant values are in [bold].














Finger phalanx blocks						
Sample code		FP-0.1 Sq	FP-0.1 Pa	FP-0.17 C25	FP-0.075 C26.5	FP-0.17 C30
Weight	(gm)	12.25	11.43	15.03	9.95	13.60
Deformation	(mm)	17.5	14	17.5	18.55	21
E_{dd}	(J/Inch ³)	34	37	41	16	26
Spine (Vertebrae) blocks						
Sample code		SV8-0.1 Sq	SV8-0.1 Pa	SV8-0.17 C25	SV8-0.17 C30	
Weight	(gm)	13.41	14.15	16.88	14.70	
Deformation	(mm)	17.5	14	17.5	21	
E_{dd}	(J/Inch ³)	27	34	36	24	
Ear canal blocks						
Sample code		EC8-0.1 Sq	SV8-0.075 C26.5	EC8-0.17 C30		
Weight	(gm)	13.37	11.74	15.21		
Deformation	(mm)	17.5	18.55	21		
E_{dd}	(J/Inch ³)	24	14	27		
Pure aluminum foam block						
Sample code		Pure—Sq				
Weight	(gm)	5.91				
Deformation	(mm)	17.5				
E_{dd}	(J/Inch ³)	21				

Table 8. All blocks energy dissipation density and impact force results for 1 inch³ volume and 70% strain.

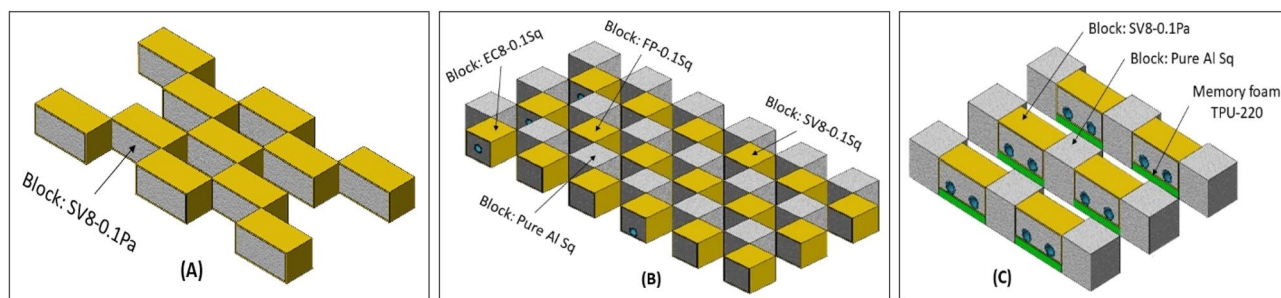


Figure 28. Categories of ACCFBs patterns (a) Similar, (b) Multi blocks, (c) Combined.

Total covered area = $(12 \times 2.5) \times (6 \times 2.5) = 450 \text{ cm}^2$	Height of pattern: 2.5 cm²
Pure Al blocks $E_{dd} = 18 \times 21 = 252 \text{ J}$	Mass = $18 \times 5.9 = 106.2 \text{ gm}$
FP-0.1Sq blocks $E_{dd} = 12 \times 34 = 408 \text{ J}$	Mass = $12 \times 12.25 = 147 \text{ gm}$
SV8-0.1Sq blocks $E_{dd} = 3 \times 27 = 81 \text{ J}$	Mass = $3 \times 13.41 = 40.23 \text{ gm}$
EC8-0.1Sq blocks $E_{dd} = 3 \times 24 = 72 \text{ J}$	Mass = $3 \times 13.37 = 40.11 \text{ gm}$
Total pattern $E_{dd} = 813 \text{ J}$	Total pattern mass = 333.5 gm

Table 9. Calculations of total energy absorption, covered area and mass of pattern (b) in Fig. 28.

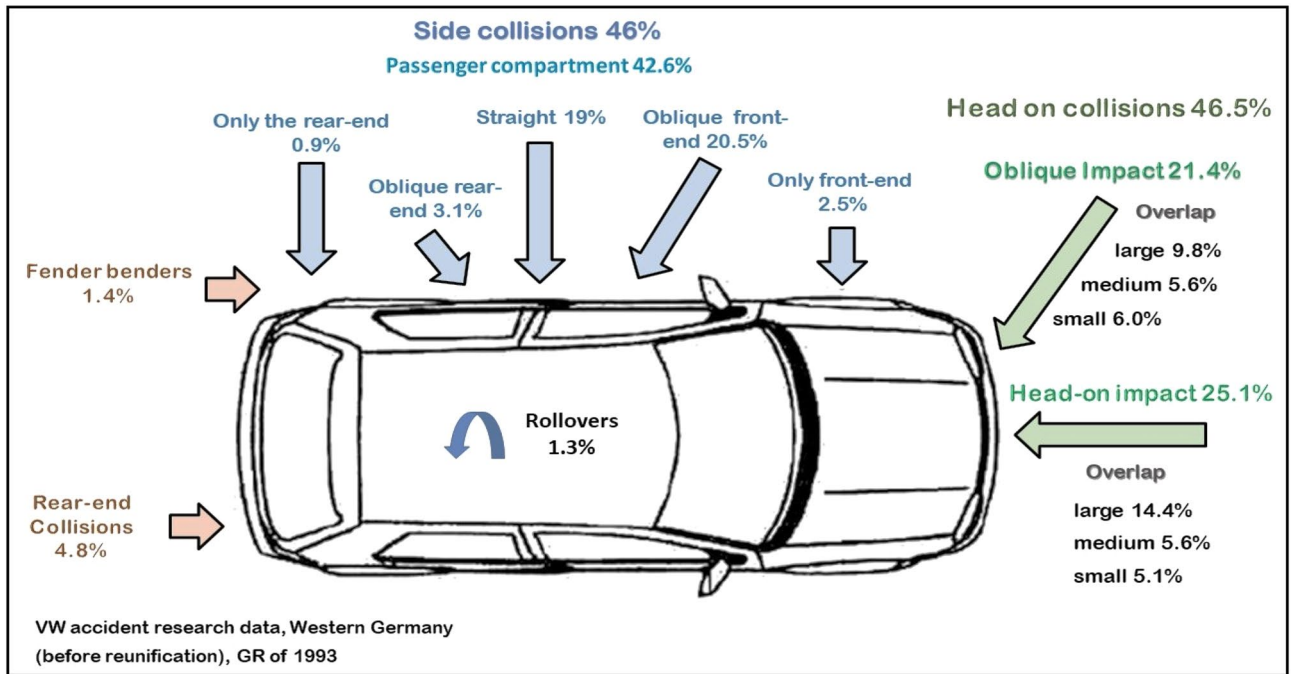


Figure 29. Evaluation of the collision types by Volkswagen company presented on 1993.

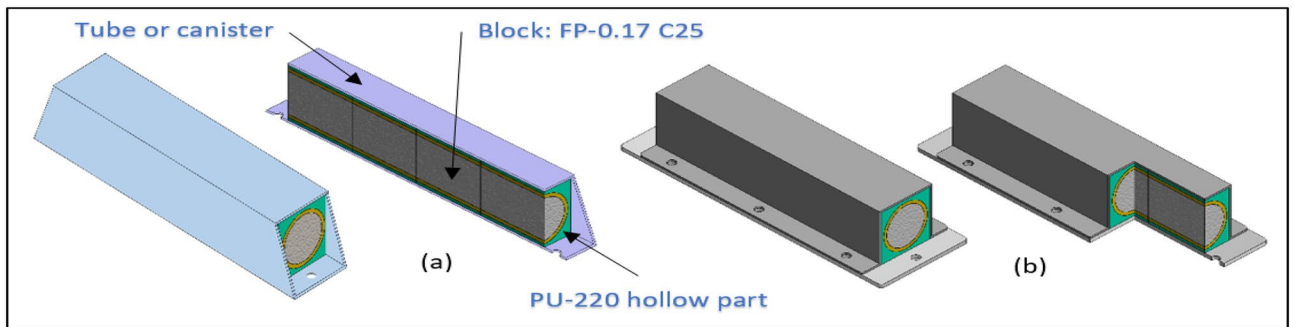


Figure 30. Filled tubes by blocks (a) one part squared tube, (b) Two parts squared tube (C shape + flat strut).

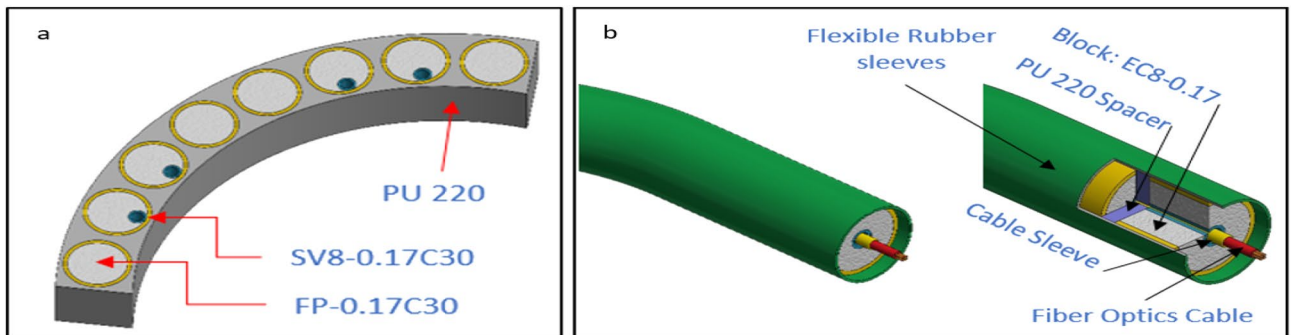


Figure 31. Flexible pattern (a) Strand for nonuniform shapes, (b) flex shield for guarding wires like fibre optic.

Figure 30 exposes examples of two patterns for filled tubes which depend on the use of highest value of energy absorption for finger phalanx block (FP-0.17 C25) with PU-220 as cushioning material to make slight damping and absorbing energy and make block easy in and easy out in fixation. After fixing the block in the PU-220 which has square shape with segment length of 28 mm block will put 4 parts with total length 128 mm in canister or tube with dimensions $160 \times 30 \times 30 \text{ mm}^3$ and thickness 1 mm. A lot of different shapes of canisters or tubes which can consist of 1 part tube or 2 parts assembled together. Length of tube and numbers of used blocks and

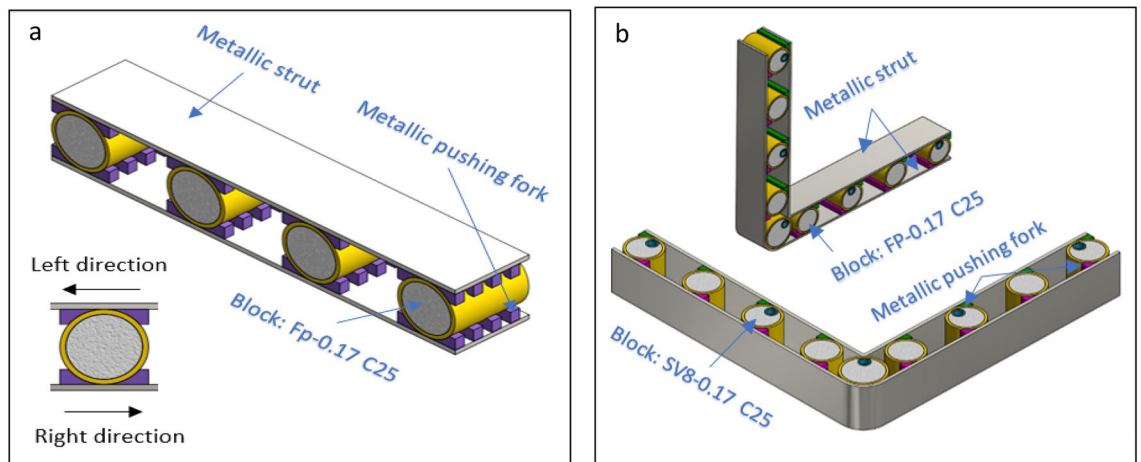


Figure 32. (a) Pattern work in two directions (b) Pattern work in two plans.

PU-220 shield could be selected according to required length or application. This filled tube part is suitable for car's sills front and rear bumpers, frames of buses and pickup cars. It is easy for common man to produce it and fix it. It is easy to calculate the total energy absorption, mass and area of this part after design. This filled tube E_{dd} equals twice E_{dd} of pure Al foam.

Figure 31 displays examples of two flexible patterns one for nonuniform shapes and one for guarding expensive wires. The first one is designed to be able to take any profile shape and different blocks can be used also can cover by belt strands or adhesive with chain strands for heavy applications. The second which guard wires like fibre optic bending angle could be controlled through flexible spacer from PU-220 or soft silicon through using maximum strain for estimate bending angle also cover material stretching should be calculated to avoid passing critical bending angle.

Figure 32 displays examples of two double-motion effect patterns the first one is designed to treat with oblique impact where it is able to work in two directions left and right with high efficiency in using all blocks ability to absorb energy. The second one is designed to improve the efficiency of energy absorption by absorbing energy in two perpendicular planes. They are suitable for protected vehicles with big mass (i.e., pickup cars and trucks) or slow vehicles for heavy work and susceptible to overturn or collisions due to their work in limited areas like forklifts. Patterns can be supported with additional blocks in their spaces but it will work in one direction or one plan also, cooling shouldn't be considered for equipment where more foam means more heat isolation.

Finally, the ACCFBs properties can be tailored through material selection for metallic foam and shield tubes dimensions and density but should take into consideration for blocks and patterns corrosion potentials of materials to avoid corrosion, heat transfer, working medium and the best distribution for blocks in pattern to take maximum benefits for mass, energy absorption and covered area.

Conclusions

- Aluminum foam blocks energy absorption has been enhanced due to shielding it by traditional aluminum foam tubes.
- Inner tube diameter should not exceed 1/3 height or diameter of the block to save foam properties as leading properties for energy absorption.
- Block selected to be with optimum designed volume 1 cubic inch to suit all types of foam also to be easy in comparing with other blocks dimensions and energy absorption.
- Each block has its energy absorption and impact force value which give variety for use in different applications, especially for vehicles.
- The compressive strength is directly proportional to the energy absorption.
- The highest value for energy absorption belongs to thick wall cylinder shapes with small diameters (25 mm, $t = 1.7$ mm) while the lowest value is belonging thin wall cylinder shapes (26.5 mm, $t = 0.75$ mm) which is less than Al foam cube by about 30%.
- The lowest impact force belongs to thin wall circular shapes and the highest belongs to the finger phalanx parallelogram block. The best properties belonging the circular thick wall blocks with large diameter (30 mm, $t = 1.7$) where it has high energy absorption and relatively low impact energy.
- Impact force can be used as index for selecting blocks where block has high impact force can be used as a guard for expensive wires like spine and ear canal blocks while block has low impact force will be suitable to absorb energy by high amount which suits for armoring against impacts.

Data availability

All data generated or analyzed during this study are included in this published article.

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References

1. Innovative solutions made from aluminum foam for the mechanical engineering sector catalogue, “Havel Metal Fam Company”.
2. Banhart, J. *Metal Foams from Fundamental Research to Applications* (Universities Press (India) Limited, 2005).
3. Wang, G., Zhang, Y., Zheng, Z., Chen, H. & Yu, J. *Crashworthiness Design and Impact Tests of Aluminum Foam-Filled Crash Boxes* (Elsevier, 2022).
4. Nakajima, H. *Porous Metals with Directional Pores* (Springer, 2013).
5. Xie, S. J., Lu, Y., Yoon, S., Yang, J. & Park, D. S. Intensity variation normalization for finger vein recognition using guided filter based single scale retinex. *Sensors* **15**(7), 17089–17105 (2015).
6. Marieb, E. & Hoehn, K. N. *Human Anatomy & Physiology* 225–239 (Global Edition-Pearson Higher Education, 2016).
7. Algabary, K. M. S., Omar, K. & Nordin, M. J. 3-dimensional ear recognition based iterative closest point with stochastic clust. *J. Comput. Sci.* **10**(3), 477 (2014).
8. Ashby, M., Evans, A. & Fleck, N. *Metal Foams: A Design Guide* 40 (Elsevier, 2000).
9. Matijašević, B. *Characterization and Optimization of Blowing Agent for Making Improved Metal Foams* (Technischen Universität Berlin, 2006).
10. Laughlin, D. E. & Hono, K. *Physical Metallurgy* 5th edn, Vol. 3, 2496–2500 (Elsevier, 2014).
11. Rajak, D. K., Kumaraswamidhas, L. A., Das, S. Technical overview of aluminum alloys foam. *Adv. Mater. Sci.* (2017).
12. Banhart, J. Aluminium foams for lighter vehicles. *Int. J. Veh. Des.* **37**(2–3), 114–125 (2005).
13. European Standard - EN 755-2 Aluminum and aluminum alloys—Extruded rod/bar, tube and profiles—Part 2: Mechanical properties (2016).
14. Zhang, J., An, Y. & Ma, H. Research progress in the preparation of aluminum foam composite structures. *Met. J. MDPI* **2**(12), 2047 (2022).
15. Šimančík, F., Rajner, W., Jerz, J. ALULIGHT - Highly porous foamed aluminium panels with outstanding properties at low density, in *European Automotive Congress Bratislava, 18th - 20th June* (2001).
16. Hamdy, A. M., Dadoura, H. H., Rabeeh, B. M., Ali, S. H. R. *Design and Response of Advanced Composite Material for Automotive Applications*. Master thesis, Helwan University (2010).
17. Budynas, R. & Nisbett, K. *Shigley's Mechanical Engineering Design* 11th edn, 93–122 (McGraw Hill Education, 2019).
18. Beer, F. Jr., Johnston, E. R., Wolf, J. & Mazurek, D. *Mechanics of Materials* 273–285 (McGraw-Hill Education, 2014).
19. Ashby, M. F. *Materials Selection in Mechanical Design* 5th edn, 494–549 (Elsevier, 2016).
20. Testing of Metallic Materials - Compression Test of Metallic Cellular Materials: “DIN 50134:2008–10” standard (2008).
21. Fazita, M. N., Khalil, H. A., Izzati, A. N. A. & Rizal, S. Effects of strain rate on failure mechanisms and energy absorption in polymer composites. In *Failure Analysis in Biocomposites, Fibre-Reinforced Composites and Hybrid Composites* 51–78 (Woodhead Publishing, 2019).
22. Niebylski, L. M., Fanning, R. J. *Metal Foams as Energy Absorbers for Automobile Bumpers*. *SAE Transactions*, **81**, Section 3: *Papers 720447–720743*, 1676–1682 (SAE International, 1972).
23. Haidar, S., Ansary, S., Rahman, A. Production and compressive characterization of aluminum MMC foam manufactured using dual foaming agent, in *IOP 5th National Conference on Processing and Characterization of Materials: Materials Science and Engineering*, 115 (2016).
24. Kader, M. A. *et al.* Macro and micro collapse mechanisms of closed-cell aluminium foams during quasi-static compression. *Mater. Des.* **118**, 11–21 (2017).
25. Gupta, N. K. & Abbas, H. Lateral collapse of composite cylindrical tubes between flat platens. *Int. J. Impact Eng.* **24**, 329–346 (2000).
26. Gupta, N. K., Sekhon, G. S. & Gupta, P. K. A study of lateral collapse of square and rectangular metallic tubes. *Thin Walled Struct.* **39**, 745–772 (2001).
27. Srivastava, V. C. & Sahoo, K. L. Processing, stabilization and applications of metallic foams. *Art of science. Mater. Sci. -Pol.* **25**(3), 733–754 (2007).
28. Ramirez, B. J. & Gupta, V. Energy absorption and low velocity impact response of open-cell polyurea foams. *Jo. Dyn. Behav. Mater.* **5**, 132–142 (2019).

Author contributions

Conceptualization, M. H. Dadoura, A. I. Farahat, M. R. Taha, and R. N. Elshaer; methodology, M. H. Dadoura, A. I. Farahat, and R. N. Elshaer; validation, M. H. Dadoura, A. I. Farahat, and R. N. Elshaer; formal analysis, M. H. Dadoura, and R. N. Elshaer; investigation, M. H. Dadoura, and R. N. Elshaer; resources, M. H. Dadoura and A. I. Farahat; data curation, M. H. Dadoura, A. I. Farahat, and R. N. Elshaer; writing-original draft preparation, M. H. Dadoura, A. I. Farahat, M. R. Taha, and R. N. Elshaer; writing-review and editing, M. H. Dadoura, A. I. Farahat, M. R. Taha, and R. N. Elshaer; visualization, M. H. Dadoura, A. I. Farahat, and R. N. Elshaer; supervision, M. H. Dadoura, A. I. Farahat, M. R. Taha, and R. N. Elshaer. All authors have read and agreed to the published version of the manuscript.

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Competing interests

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Additional information

Correspondence and requests for materials should be addressed to M.H.D. or R.N.E.

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